Finite Volume Methods for Hyperbolic Problems

Linear Hyperbolic Systems

- · General form, coefficient matrix, hyperbolicity
- Scalar advection equation
- Linear acoustics equations
- Eigen decomposition
- Characteristics and general solution
- Boundary conditions

Linear hyperbolic systems

Linear system of *m* equations: $q(x,t) \in \mathbb{R}^m$ for each (x,t) and

$$q_t + Aq_x = 0, \qquad -\infty < x, \infty, \ t \ge 0.$$

A is $m \times m$ matrix (constant for now, independent of x, t)

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Eigenvalues are wave speeds.

Eigenvectors used to split arbibrary data into waves. So matrix of eigenvectors must be nonsingular.

Advection equation as a linear system

 $q_t + uq_x = 0$

with *u* a constant (real) velocity. $(1 \times 1 \text{ diagonalizable}, \lambda^1 = u)$ Initial condition:

$$q(x,0) = \overset{\circ}{q}(x), \quad -\infty < x < \infty.$$

The solution to this Cauchy problem is:

$$q(x,t) = \overset{\mathsf{o}}{q}(x-ut)$$

It is constant along each characteristic curve

$$X(t) = x_0 + ut$$

Characteristics for advection

$$q(x,t) = \stackrel{\circ}{q}(x-ut) \implies q$$
 is constant along lines

$$X(t) = x_0 + ut, \quad t \ge 0.$$

Can also see that q is constant along X(t) from:

$$\frac{d}{dt}q(X(t),t) = q_x(X(t),t)X'(t) + q_t(X(t),t) = q_x(X(t),t)u + q_t(X(t),t) = 0.$$

In x-t plane:



Consider constant coefficient linear system $q_t + Aq_x = 0$.

Suppose hyperbolic:

- Real eigenvalues $\lambda^1 \leq \lambda^2 \leq \cdots \leq \lambda^m$,
- Linearly independent eigenvectors r^1, r^2, \ldots, r^m .

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Let $R = [r^1 | r^2 | \cdots | r^m]$ $m \times m$ matrix of eigenvectors.

Then $Ar^p = \lambda^p r^p$ means that $AR = R\Lambda$ where

$$\Lambda = \begin{bmatrix} \lambda^1 & & & \\ & \lambda^2 & & \\ & & \ddots & \\ & & & \lambda^m \end{bmatrix} \equiv \operatorname{diag}(\lambda^1, \lambda^2, \dots, \lambda^m).$$

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 $AR = R\Lambda \implies A = R\Lambda R^{-1}$ and $R^{-1}AR = \Lambda$. Similarity transformation with *R* diagonalizes *A*.

Consider constant coefficient linear system $q_t + Aq_x = 0$. Multiply system by R^{-1} :

$$R^{-1}q_t(x,t) + R^{-1}Aq_x(x,t) = 0.$$

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Use $R^{-1}AR = \Lambda$ and define $w(x,t) = R^{-1}q(x,t)$:

 $w_t(x,t) + \Lambda w_x(x,t) = 0.$ Since *R* is constant!

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 $w_t(x,t) + \Lambda w_x(x,t) = 0.$ Since *R* is constant!

This decouples to *m* independent scalar advection equations:

$$w_t^p(x,t) + \lambda^p w_x^p(x,t) = 0.$$
 $p = 1, 2, ..., m$

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The solution to the decoupled equation $w_t^p + \lambda^p w_x^p = 0$ is

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Putting these together in vector gives w(x, t) and finally

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We can rewrite this as

$$q(x,t) = \sum_{p=1}^{m} w^{p}(x,t) r^{p} = \sum_{p=1}^{m} w^{0}(x-\lambda^{p}t) r^{p}$$

Linear acoustics

Example: Linear acoustics in a 1d gas tube

$$q = \left[\begin{array}{c} p \\ u \end{array} \right] \qquad \begin{array}{c} p(x,t) = \text{pressure perturbation} \\ u(x,t) = \text{velocity} \end{array}$$

Equations:

 $p_t + K_0 u_x = 0$ Change in pressure due to compression $\rho_0 u_t + p_x = 0$ Newton's second law, F = ma

where K_0 = bulk modulus, and ρ_0 = unperturbed density. Hyperbolic system:

$$\left[\begin{array}{c}p\\u\end{array}\right]_t+\left[\begin{array}{cc}0&K_0\\1/\rho_0&0\end{array}\right]\left[\begin{array}{c}p\\u\end{array}\right]_x=0.$$

Eigenvectors for acoustics

$$A = \left[\begin{array}{cc} 0 & K_0 \\ 1/\rho_0 & 0 \end{array} \right]$$

Eigenvectors:

$$r^1 = \begin{bmatrix} -\rho_0 c_0 \\ 1 \end{bmatrix}, \qquad r^2 = \begin{bmatrix} \rho_0 c_0 \\ 1 \end{bmatrix}.$$

Check that $Ar^p = \lambda^p r^p$, where

$$\lambda^1 = -c_0, \qquad \lambda^2 = +c_0.$$

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Physical meaning of eigenvectors

Eigenvectors for acoustics:

$$r^{1} = \begin{bmatrix} -\rho_{0}c_{0} \\ 1 \end{bmatrix} = \begin{bmatrix} -Z_{0} \\ 1 \end{bmatrix}, \qquad r^{2} = \begin{bmatrix} \rho_{0}c_{0} \\ 1 \end{bmatrix} = \begin{bmatrix} Z_{0} \\ 1 \end{bmatrix}$$

Consider a pure 1-wave (simple wave), at speed $\lambda^1 = -c_0$, If $\overset{\circ}{q}(x) = \bar{q} + \overset{\circ}{w}^1(x)r^1$ then

$$q(x,t) = \bar{q} + \overset{\circ}{w}^{1}(x - \lambda^{1}t)r^{1}$$

Variation of q, as measured by q_x or $\Delta q = q(x + \Delta x) - q(x)$ is proportional to eigenvector r^1 , e.g.

$$q_x(x,t) = \overset{\circ}{w}^1_x(x-\lambda^1 t)r^1$$

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In a simple 1-wave (propagating at speed $\lambda^1 = -c_0$),

$$\left[\begin{array}{c} p_x \\ u_x \end{array}\right] = \beta(x) \left[\begin{array}{c} -Z_0 \\ 1 \end{array}\right]$$

The pressure variation is $-Z_0$ times the velocity variation.

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Similarly, in a simple 2-wave ($\lambda^2 = c_0$),

$$\left[\begin{array}{c} p_x \\ u_x \end{array}\right] = \beta(x) \left[\begin{array}{c} Z_0 \\ 1 \end{array}\right]$$

The pressure variation is Z_0 times the velocity variation.

$$q(x,0) = \begin{bmatrix} p(x) \\ 0 \end{bmatrix} = -\frac{p(x)}{2Z} \begin{bmatrix} -Z \\ 1 \end{bmatrix} + \frac{p(x)}{2Z} \begin{bmatrix} Z \\ 1 \end{bmatrix}$$
$$= w^1(x,0)r^1 + w^2(x,0)r^2$$
$$= \begin{bmatrix} p(x)/2 \\ -p(x)/(2Z) \end{bmatrix} + \begin{bmatrix} p(x)/2 \\ p(x)/(2Z) \end{bmatrix}.$$

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The general solution for acoustics:

$$q(x,t) = w^{1}(x - \lambda^{1}t, 0)r^{1} + w^{2}(x - \lambda^{2}t, 0)r^{2}$$
$$= w^{1}(x + c_{0}t, 0)r^{1} + w^{2}(x - c_{0}t, 0)r^{2}$$

Recall that $w(x,0) = R^{-1}q(x,0)$, i.e.

$$w^{1}(x,0) = \ell^{1}q(x,0), \qquad w^{2}(x,0) = \ell^{2}q(x,0)$$

where ℓ^1 and ℓ^2 are rows of $R^{-1}.$

$$R^{-1} = \left[\begin{array}{c} \ell^1 \\ \ell^2 \end{array} \right]$$

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$$R^{-1} = \left[\begin{array}{c} \ell^1 \\ \ell^2 \end{array} \right]$$

Note: ℓ^1 and ℓ^2 are left-eigenvectors of *A*:

$$\ell^p A = \lambda^p \ell^p$$
 since $R^{-1} A = \Lambda R^{-1}$.

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The general solution for acoustics:

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Linear acoustics

Example: Linear acoustics in a 1d gas tube, linearized about $p = p_0$, $u = u_0$

$$q = \left[\begin{array}{c} p \\ u \end{array} \right] \qquad p(x,t) = \text{pressure perturbation} \\ u(x,t) = \text{velocity perturbation}$$

Equations include advective transport at speed u_0 :

 $p_t + u_0 p_x + K_0 u_x = 0$ Change in pressure due to compressio $\rho_0 u_t + p_x + u_0 u_x = 0$ Newton's second law, F = ma

where K_0 = bulk modulus, and ρ_0 = unperturbed density. Hyperbolic system:

$$\left[\begin{array}{c}p\\u\end{array}\right]_t + \left[\begin{array}{cc}u_0 & K_0\\1/\rho_0 & u_0\end{array}\right] \left[\begin{array}{c}p\\u\end{array}\right]_x = 0.$$

$$A = \left[\begin{array}{cc} u_0 & K_0 \\ 1/\rho_0 & u_0 \end{array} \right]$$

Eigenvectors:

$$r^1 = \begin{bmatrix} -\rho_0 c_0 \\ 1 \end{bmatrix}, \qquad r^2 = \begin{bmatrix} \rho_0 c_0 \\ 1 \end{bmatrix}.$$

Check that $Ar^p = \lambda^p r^p$, where

$$\lambda^1 = u_0 - c_0, \qquad \lambda^2 = u_0 + c_0.$$

with $c_0 = \sqrt{K_0/\rho_0} \implies K_0 = \rho_0 c_0^2$.

Eigenvectors for acoustics

$$A = \begin{bmatrix} u_0 & K_0 \\ 1/\rho_0 & u_0 \end{bmatrix} \qquad = u_0 \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} + \begin{bmatrix} 0 & K_0 \\ 1/\rho_0 & 0 \end{bmatrix}$$

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Note: Eigenvectors are independent of u_0 .

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Initial-boundary value problem (IBVP) for advection

Advection equation on finite 1D domain:

$$q_t + uq_x = 0 \qquad \mathbf{a} < \mathbf{x} < \mathbf{b}, \ t \ge 0,$$

with initial data

$$q(x,0) = \eta(x) \qquad a < x < b.$$

and boundary data at the inflow boundary:

If u > 0, need data at x = a:

$$q(a,t) = g(t), \qquad t \ge 0,$$

If u < 0, need data at x = b:

$$q(b,t) = g(t), \qquad t \ge 0,$$

Characteristics for IBVP

In x-t plane for the case u > 0:



Solution:

$$q(x,t) = \begin{cases} \eta(x-ut) & \text{ if } a \le x-ut \le b, \\ g((x-a)/u) & \text{ otherwise }. \end{cases}$$

Periodic boundary conditions

 $q(a,t)=q(b,t), \qquad t\geq 0.$

In *x*–*t* plane for the case u > 0:



Solution:

$$q(x,t) = \eta(X_0(x,t)),$$

where $X_0(x,t) = a + mod(x - ut - a, b - a)$.

Linear acoustics — characteristics

$$q(x,t) = w^{1}(x+ct,0)r^{1} + w^{2}(x-ct,0)r^{2}$$
$$= \frac{-\overset{\circ}{p}(x+ct)}{2Z_{0}} \begin{bmatrix} -Z_{0} \\ 1 \end{bmatrix} + \frac{\overset{\circ}{p}(x-ct)}{2Z_{0}} \begin{bmatrix} Z_{0} \\ 1 \end{bmatrix}.$$

For IBVP on a < x < b, must specify one incoming boundary condition at each side: $w^2(a, t)$ and $w^1(b, t)$

Acoustics boundary conditions



Outflow (non-reflecting, absorbing) boundary conditions:

$$w^2(a,t) = 0, \qquad w^1(b,t) = 0.$$

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Periodic boundary conditions:

$$w^2(a,t) = w^2(b,t), \qquad w^1(b,t) = w^1(a,t),$$

or simply

$$q(a,t) = q(b,t).$$

Solid wall (reflecting) boundary conditions:

$$u(a,t) = 0,$$
 $u(b,t) = 0.$

which can be written in terms of characteristic variables as:

$$w^{2}(a,t) = -w^{1}(a,t), \qquad w^{1}(b,t) = -w^{2}(a,t)$$

since $u = w^1 + w^2$.

$$q(a,t) = w^{1}(a,t) \begin{bmatrix} -Z_{0} \\ 1 \end{bmatrix} + w^{2}(a,t) \begin{bmatrix} Z_{0} \\ 1 \end{bmatrix}$$

$$\begin{bmatrix} p(a,t)\\ u(a,t) \end{bmatrix} = \begin{bmatrix} (-w^1(a,t) + w^2(a,t))Z_0\\ w^1(a,t) + w^2(a,t) \end{bmatrix} = \begin{bmatrix} -2w^1(a,t)Z_0\\ 0 \end{bmatrix}.$$

Figure 3.1 illustrates the acoustics solution with $u(x, 0) \equiv 0$. An animation can be found in the Clawpack Gallery Gallery of fymbook applications \longrightarrow Chapter 3

 \longrightarrow animation of Pressure and Velocity

Shows solution computed numerically on a fine grid, with:

- · Solid wall boundary condition at the left,
- Outflow boundary condition at the right.