

V.3

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To calculate the drag we need to

know p on $r=a$

- flow is steady, inviscid + irrotational ($\gamma=0$)

so Bernoulli Eq. = const. everywhere

$$\text{so } \frac{1}{2} \rho_0 \underline{u} \cdot \underline{u} + p = \frac{1}{2} \rho_0 U^2 + p_\infty$$

$$\text{so } p = p_\infty + \frac{1}{2} \rho_0 U^2 - \frac{1}{2} \rho_0 \underline{u} \cdot \underline{u}$$

$$\text{on } r=a \quad \underline{u} = (0, u_\theta) \quad \text{since } u_R = 0$$

$$\text{and } (u_R, u_\theta) = \nabla \phi = \left(\frac{\partial \phi}{\partial r}, \frac{1}{r} \frac{\partial \phi}{\partial \theta} \right)$$

$$\Rightarrow u_\theta = \frac{1}{r} \frac{\partial \phi}{\partial \theta} = -U \left(1 + \frac{a^2}{r^2} \right) \sin \theta = -2U \sin \theta \quad \text{on } r=a$$

$$\text{and } \left. \frac{1}{2} \rho_0 U^2 - \frac{1}{2} \rho_0 \underline{u} \cdot \underline{u} \right|_{r=a} = \frac{1}{2} \rho_0 (U^2 - u_\theta^2)$$

$$p = p_{\infty} + \frac{1}{2} \rho_0 \left[U^2 - 4U^2 \sin^2 \theta \right]$$

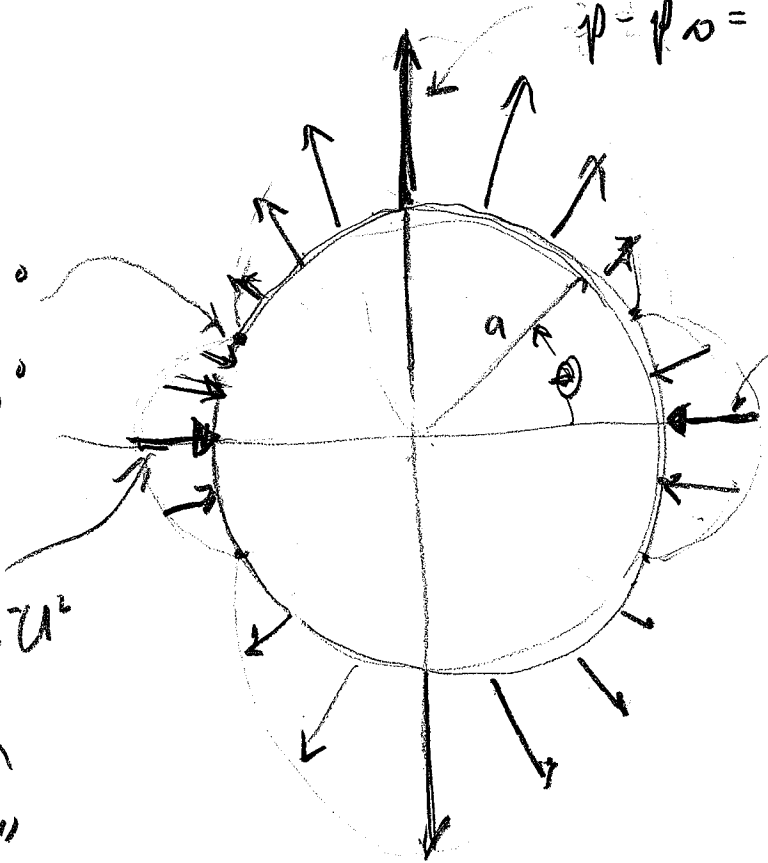
$$p = p_{\infty} + \frac{1}{2} \rho_0 U^2 [1 - 4 \sin^2 \theta]$$

$$p - p_{\infty} = -3 \left(\frac{1}{2} \rho_0 U^2 \right)$$

$p - p_{\infty} = 0$
 at $|\theta| = 30^\circ$
 & $|\theta - \pi| = 30^\circ$

$$p - p_{\infty} = \frac{1}{2} \rho_0 U^2$$

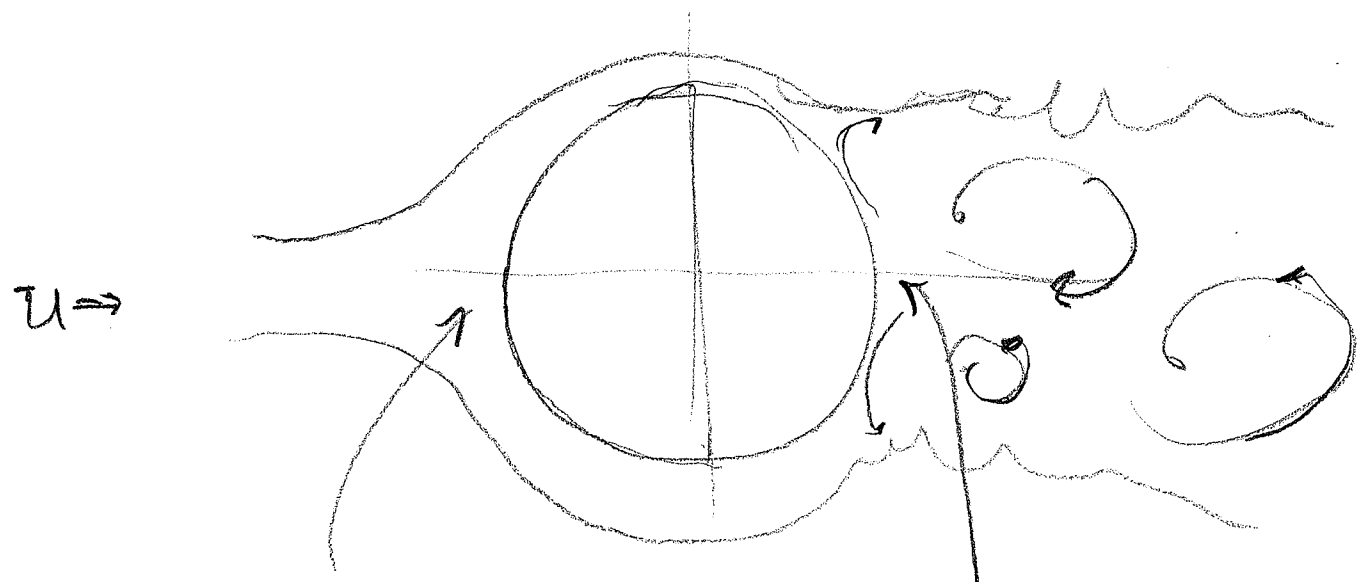
"stagnation pressure"



Pressure forces are symmetric

so drag = 0

For flow past a real cylinder
there is "separation"



$p \approx p_{\infty} + \frac{1}{2} \rho_0 U^2$
on the front
face

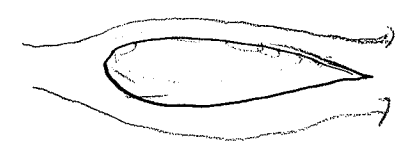
$p \approx p_0$ on the
back face

Drag $\sim \frac{1}{2} \rho_0 U^2 \cdot A$ ($A =$ frontal area)

More generally:

Drag $= \frac{1}{2} \rho_0 U^2 C_D A$

↑ a "drag coefficient"



$C_D = O(1)$ for the cylinder $C_D \ll 1$ for streamlined shapes