

ME 356 Fall 2001

Solutions to Homework #1

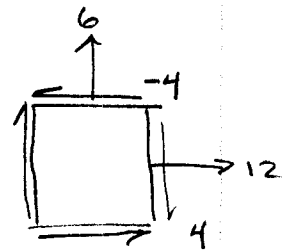
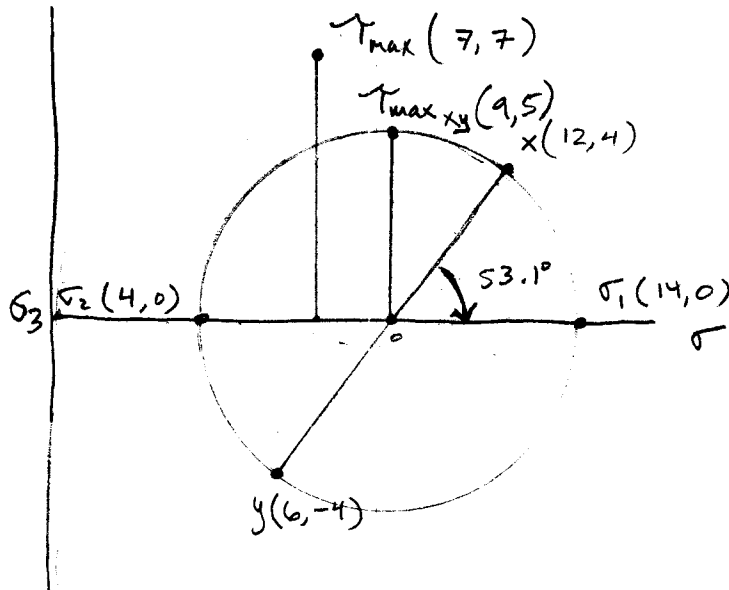
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Questions, comments, gripes, and most importantly praise should be directed towards your loving TA during office hours MTF.

3.1 a) $\sigma_x = 12$, $\sigma_y = 6$, $\tau_{xy} = 4$ CCW

There are many different ways to do Mohr's circle. This is one example.

$x(\sigma_x, -\tau_{xy})$
 $y(\sigma_y, \tau_{xy})$



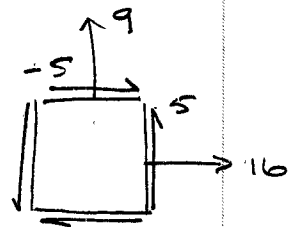
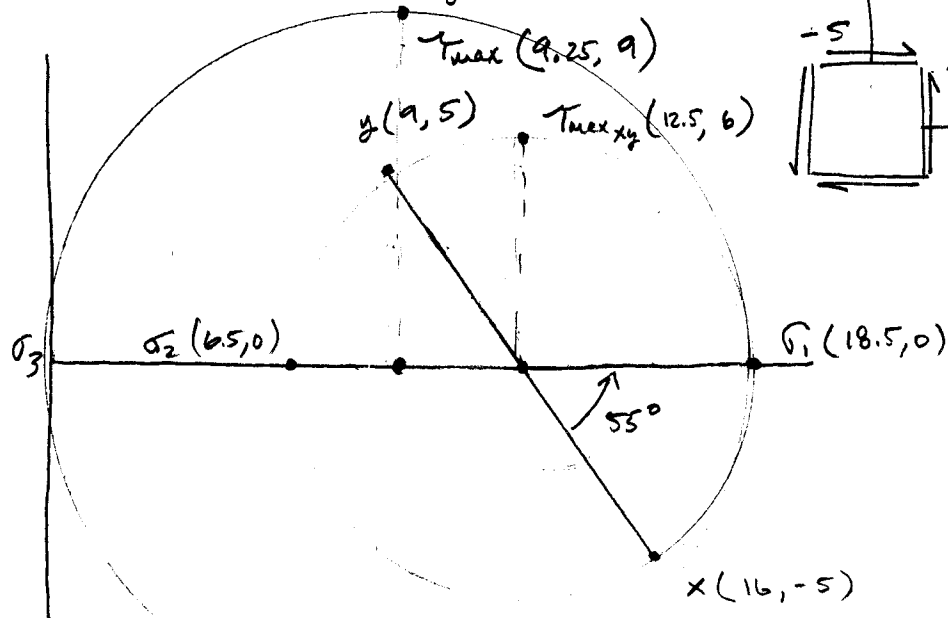
$\sigma_1 = 14$ @ $53.1/2 = 26.6^\circ$ CCW From Axis

$\sigma_2 = 4$ @ $26.6^\circ + 90^\circ = 116.6^\circ$ CCW From Axis

$\tau_{max_{xy}} = 5$ @ $26.6^\circ - 45^\circ = 18.4^\circ$ CCW From Axis

$\tau_{max_{total}} = 7$

b) $\sigma_x = 16$, $\sigma_y = 9$, $\tau_{xy} = 5$ CCW



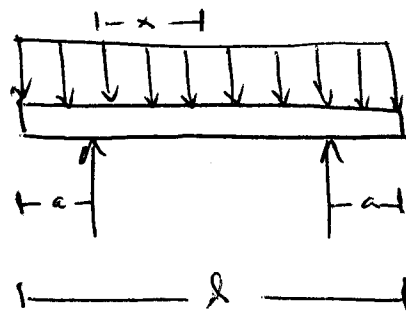
$\sigma_1 = 18.5$ @ $55/2 = 27.5^\circ$ CCW From Axis

$\sigma_2 = 6.5$ @ $27.5^\circ - 90^\circ = -62.5^\circ$ CCW From Axis

$\tau_{max_{xy}} = 6$ @ $-27.5^\circ + 45^\circ + 90^\circ = 107.5^\circ$ CCW From Axis

$\tau_{max_{total}} = 9.25$

3.13) (a)



$$M = \frac{w}{2} (lx - (a+x)^2)$$

FIND MOMENT ABOUT CENTER:

$$\begin{aligned} M_c &= \frac{wl}{2} \left(\frac{l}{2} - a \right) - \frac{wl}{2} \left(\frac{l}{4} \right) \\ &= \frac{wl}{2} \left(\frac{l}{2} - a - \frac{l}{4} \right) \\ &= \frac{wl}{2} \left(\frac{l}{4} - a \right) \Rightarrow \boxed{125 \text{ (lb} \cdot \text{in)}} \end{aligned}$$

FIND MOMENT ABOUT LEFT REACTION:

$$M_r = wa \left(\frac{a}{2} \right) = \frac{wa^2}{2} = \boxed{253.1 \text{ (lb} \cdot \text{in)}} \quad \text{Diagram: } \begin{array}{c} \text{wa} \\ \downarrow \\ \text{---} \text{a} \text{---} \end{array} M_r$$

(b) FIND OPTIMAL PLACEMENT OF SETBACK:

$$M_r = M_c$$

$$\frac{wa^2}{2} = \frac{wl}{2} \left(\frac{l}{4} - a \right)$$

$$\frac{wa^2}{2} = \frac{wl^2}{84} - \frac{wla}{2}$$

$$a^2 = \frac{l^2}{4} - la \rightarrow \text{divide by } l^2$$

$$\frac{a^2}{l^2} = \frac{1}{4} - \frac{a}{l} \rightarrow \frac{a^2}{l^2} + \frac{a}{l} - \frac{1}{4} = 0$$

quadratic equation:

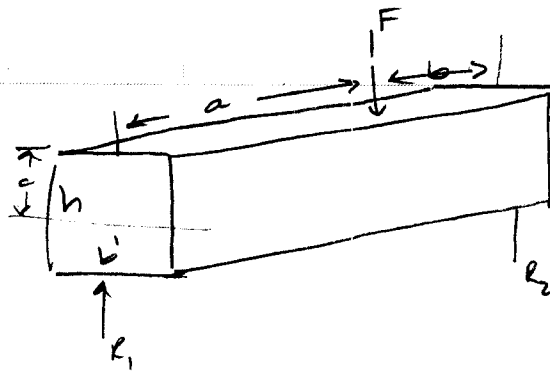
$$\frac{a}{l} = \frac{-1 \pm \sqrt{1 + 4(1)(\frac{1}{4})}}{2} = .207 \quad a = .207l$$

$$\therefore \text{ since } M_{\min} = \frac{wa^2}{2} = \frac{w(.207)^2 l^2}{2}$$

$$= .0214 wl^2 = .0214(100)(10^2)$$

$$= \boxed{214 \text{ in} \cdot \text{lb}}$$

3-20)



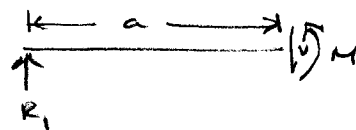
$$l = a + b$$

a) Show That $F = \frac{\sigma b' h^2 l}{6ab}$ — make correction in Book

$$\sum M_{R_2} = 0$$

$$-R_1(a+b) + F(b) = 0$$

$$R_1 = \frac{Fb}{a+b} = \frac{Fb}{l}$$



$$M = R_1 a = \frac{Fba}{l}$$

$$\sigma = \frac{Mc}{I}$$

$$I = \frac{b' h^3}{12} \quad c = h/2$$

$$= \frac{Mc}{b' h^2} = \frac{b F b a}{b' h^2 l}$$

$$\therefore F = \frac{b' h^2 \sigma l}{6ba}$$

$$b) \frac{F_m}{F} = \frac{(\sigma_m/\sigma) (b'_m/b') (h_m/h)^2 (l_m/l)}{(l_m/a) (b_m/b)}$$

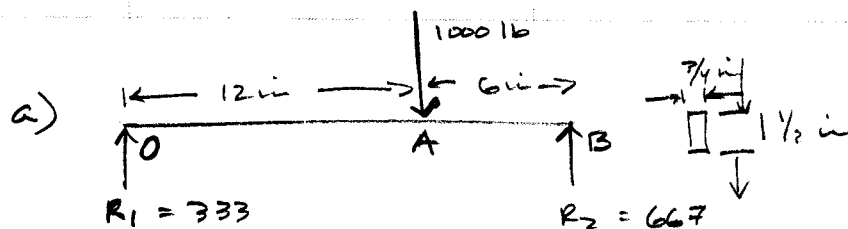
$$S = \text{Scale Factor and } \sigma_m/\sigma = 1$$

$$\frac{(1)(s)(s)^2(s)}{(s)(s)} = s^2$$

$$F_m = s^2 F_m = 2^2 F_m = 4F$$

ALTHOUGH WE'VE DOUBLED THE PHYSICAL DIMENSIONS WE ONLY QUADRUPLD THE LOAD CARRYING ABILITY.

3-26)



$$\sum M_{R_2} = 0$$

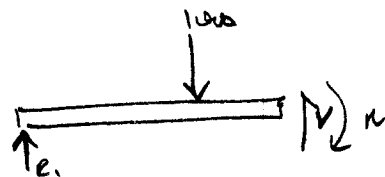
$$-R_1(18) + 1000(6) = 0$$

$$R_1 = 333 \text{ lb}$$

$$\sum F = 0$$

$$333 - 1000 + V = 0$$

$$V = 667$$



$$\tau_{\max} = \frac{3V}{2A} = \frac{3(667)}{2(1.25)} = \boxed{890 \text{ psi}}$$

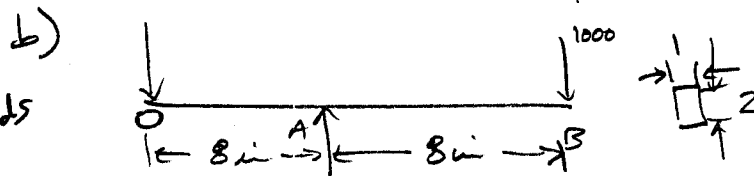
⇒ located on neutral axis b/w points A and B

$$\sigma_{\max} = \frac{Mc}{I} \quad M_A = M_{\max} = 333(12) = 4000 \text{ lb}\cdot\text{in}$$

$$= \frac{4000(.75)}{.211} \quad c = h/2 = .75$$

$$= \boxed{14218 \text{ psi}} \quad I = \frac{bh^3}{12} = \frac{3/4(3/2)^3}{12} = .211$$

located @ point A



Use same methods
As in part A
For calculations

$$\tau_{\max} = \frac{3(1000)}{2(2)} = \boxed{750 \text{ psi}} \quad \text{on neutral surface from O to B.}$$

$$\sigma_{\max} = \frac{8000(1)}{.667} = \boxed{12,000 \text{ psi}} @ A$$

$$3-30) a) J_{\text{solid}} = \frac{\pi d^4}{32}$$

$$= \frac{\pi (7)^4}{32}$$

$$= 236 \text{ cm}^4$$

$$J_{\text{Hollow}} = \frac{\pi}{32} (d_o^4 - d_i^4)$$

$$= \frac{\pi}{32} (7^4 - 5^4)$$

$$= 125 \text{ cm}^4$$

$$\text{Percent Reduction} = \frac{236 - 125}{236} \times 100$$

$$= \boxed{47\%}$$

$$b) \text{ let } k = I \cdot e \quad \text{and } C = k \frac{\pi}{4}$$

$$W_{\text{solid}} = kA$$

$$= k \frac{\pi}{4} d^2$$

$$= C d^2$$

$$= 49C$$

$$W_{\text{Hollow}} = kA$$

$$= k \frac{\pi}{4} (d_o^2 - d_i^2)$$

$$= C (d_o^2 - d_i^2)$$

$$= C (49 - 5.8^2)$$

$$= 15.4C$$

$$\text{Percent Reduction} = \frac{49 + 15}{49} \times 100$$

$$= 68.6\%$$

3-35) Let $n = 5 \text{ rev/min}$ and $r = 3 \text{ in}$

$$\text{Then, } V = \frac{2\pi r n}{12} = \frac{2\pi(3)(5)}{12} = 7.85 \text{ ft/min}$$

$$F = \frac{H(33000)}{V} = \frac{(1)(33000)}{7.85} \approx 4200 \text{ lb}$$

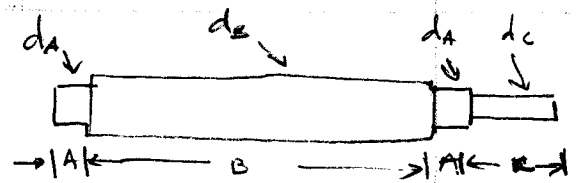
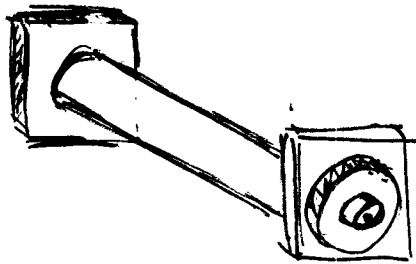
$$T = F \cdot \text{radius of cylinder}$$

$$= 4200(3) = 12600 \text{ lb}\cdot\text{in}$$

$$\tau = \frac{16T}{\pi d^3} \quad \left. \vphantom{\tau = \frac{16T}{\pi d^3}} \right\} \text{SEE solution to 36 for derivation of this EGN.}$$

$$d = \left(\frac{16(12600)}{\pi(14000)} \right)^{1/3} = \boxed{1.66 \text{ in.}}$$

3-36)



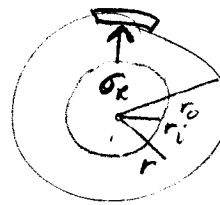
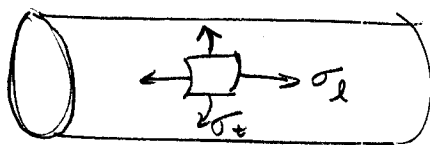
$$\omega = 2\pi n / 60 = 2\pi(8) / 60 = .838 \text{ rad/s}$$

$$T = \frac{H}{\omega} = \frac{1000}{.838} = 1190 \text{ N.m}$$

$$\tau_{\max} = \frac{T r}{J} \quad r = \frac{1}{2} d \quad J = \frac{\pi}{32} (d^4)$$

$$= \frac{T (\frac{d}{2})}{\frac{\pi}{32} d^4} \Rightarrow d = \left(\frac{16 T}{\pi \tau} \right)^{1/3} = \left(\frac{16 (1190)}{\pi (75)} \right)^{1/3} = \boxed{4.32 \text{ cm}}$$

3-41) determine whether pipe is thin or thick walled: $\frac{\text{radius}}{t} \geq 20 \Rightarrow \text{thin walled}$
 $\frac{3}{\frac{1}{4}} = 12$; so the cylinder is thick walled.



$$r_i = 2.75$$

$$r_o = 3$$

$\sigma_r, \sigma_t, \sigma_l$ are all principle stresses

$$\sigma_t = P \frac{r_o^2 + r_i^2}{r_o^2 - r_i^2} = +11.52 P$$

$$\sigma_l = \frac{P r_i^2}{r_o^2 - r_i^2} = 5.26 P$$

$$\sigma_r = P \frac{(r_i^2 - r_o^2)}{r_o^2 - r_i^2} = -P$$

3-41 continued)

$$\therefore \tau_{max} = \frac{\sigma_t - \sigma_r}{2} = \frac{(11.52 + 1)P}{2} = 6.26P$$

$$\tau_{max} = 4000 \text{ psi}$$

$$\frac{4000}{6.26} = P = \boxed{639 \text{ psi}}$$

3-44) Thick walled vessel

$$r_i = .875$$

$$r_o = 1.1875$$

$$\sigma_t = P \frac{(r_o^2 + r_i^2)}{r_o^2 - r_i^2} = 3.38P$$

$$\sigma_r = P \frac{(r_i^2 - r_o^2)}{r_o^2 - r_i^2} = -P$$

$$\sigma_t > \sigma_r > \sigma_c$$

$$\sigma_c = \frac{Pr_i^2}{r_o^2 - r_i^2} = 1.18P$$

$$\therefore \sigma_t = \sigma_{max}$$

$$P = \frac{.8(57000)}{3.38} = \boxed{13491 \text{ psi}}$$

USE TABLE E-20 TO GET YIELD STRESS OF MATERIAL

$$S_y = 57 \text{ Kpsi}$$