Conservation Laws and Finite Volume Methods AMath 574 Winter Quarter, 2011

Randall J. LeVeque

Applied Mathematics University of Washington

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Outline

Today:

- · Gas dynamics
- · Linearization of gas dynamics
- Linear acoustics
- Diagonalization of linear systems
- · Meaning of eigenvectors
- · Characteristic solution for acoustics

Next:

- · Riemann problem for acoustics
- · Finite volume methods

Reading: Chapter 3 and start Chapter 4

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Compressible gas dynamics

In one space dimension (e.g. in a pipe).

$$\rho(x,t) = \text{density}, \quad u(x,t) = \text{velocity},$$

$$p(x,t) = \text{pressure}, \quad \rho(x,t)u(x,t) = \text{momentum}.$$

Conservation of:

mass: ρ flux: ρu momentum: ρu flux: $(\rho u)u + p$ (energy)

Conservation laws:

$$\rho_t + (\rho u)_x = 0$$
$$(\rho u)_t + (\rho u^2 + p)_x = 0$$

Equation of state:

$$p = P(\rho)$$
.

(Later: p may also depend on internal energy / temperature)

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Compressible gas dynamics

Conservation laws:

$$\rho_t + (\rho u)_x = 0$$
$$(\rho u)_t + (\rho u^2 + p)_x = 0$$



Momentum flux:

$$\rho u^2 = (\rho u)u = advective flux$$

p term in flux?

- $-p_x$ = force in Newton's second law,
- as momentum flux: microscopic motion of gas molecules.

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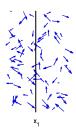
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Momentum flux arising from pressure



Note that:

- molecules with positive x-velocity crossing x_1 to right increase the momentum in $[x_1, x_2]$
- molecules with negative x-velocity crossing x_1 to left also increase the momentum in $[x_1, x_2]$

Hence momentum flux increases with pressure $p(x_1, t)$ even if macroscopic (average) velocity is zero.

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Compressible gas dynamics

Conservation laws:

$$\rho_t + (\rho u)_x = 0$$
$$(\rho u)_t + (\rho u^2 + p)_x = 0$$

Equation of state:

$$p = P(\rho)$$
.

Same as shallow water if $P(\rho) = \frac{1}{2}g\rho^2$ (with $\rho \equiv h$).

Isothermal: $P(\rho) = a^2 \rho$ (since T proportional to p/ρ).

Isentropic: $P(\rho) = \hat{\kappa} \rho^{\gamma} \ (\gamma \approx 1.4 \text{ for air})$

Jacobian matrix:

$$f'(q) = \begin{bmatrix} 0 & 1 \\ P'(\rho) - u^2 & 2u \end{bmatrix}, \qquad \lambda = u \pm \sqrt{P'(\rho)}.$$

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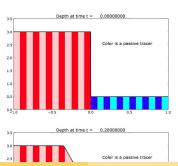
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The Riemann problem

Dam break problem for shallow water equations

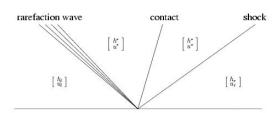
$$h_t + (hu)_x = 0$$

$$(hu)_t + (hu^2 + \frac{1}{2}gh^2)_x = 0$$



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Riemann solution for the SW equations in x-t plane



Similarity solution:

Solution is constant on any ray: q(x,t) = Q(x/t)

Riemann solution can be calculated for many problems. Linear: Eigenvector decomposition. Nonlinear: more difficult.

In practice "approximate Riemann solvers" used numerically.

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Compressible gas dynamics

Conservation laws:

$$\rho_t + (\rho u)_x = 0$$
$$(\rho u)_t + (\rho u^2 + p)_x = 0$$

Equation of state:

$$p = P(\rho)$$
.

Jacobian matrix:

$$f'(q) = \begin{bmatrix} 0 & 1 \\ P'(\rho) - u^2 & 2u \end{bmatrix}, \qquad \lambda = u \pm \sqrt{P'(\rho)}.$$

Sound speed: $c = \sqrt{P'(\rho)}$ varies with ρ .

System is hyperbolic if $P'(\rho) > 0$.

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Linearization of gas dynamics

Suppose $\rho(x,t) \approx \rho_0$ and $u(x,t) \approx u_0$.

Model small perturbations to this steady state (sound waves).

$$\left[\begin{array}{c} \rho(x,t) \\ (\rho u)(x,t) \end{array}\right] = \left[\begin{array}{c} \rho_0 \\ \rho_0 u_0 \end{array}\right] + \left[\begin{array}{c} \widetilde{\rho}(x,t) \\ (\widetilde{\rho u})(x,t) \end{array}\right]$$

or $q(x,t) = q_0 + \tilde{q}(x,t)$ where $\|\tilde{q}(x,t)\| = \epsilon$ is small

Then nonlinear equation $q_t + f(q)_x = 0$ leads to

$$\tilde{q}_t = q_t
= -f(q)_x
= -f'(q)q_x
= -f'(q_0 + \tilde{q})\tilde{q}_x
= -f'(q_0)\tilde{q}_x + \mathcal{O}(\epsilon^2).$$

Linearization: $\tilde{q}_t + A\tilde{q}_x = 0$ where $A = f'(q_0)$.

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Linearization of gas dynamics

Linearization: $\tilde{q}_t + A\tilde{q}_x = 0$ where $A = f'(q_0)$.

$$A = f'(q_0) = \begin{bmatrix} 0 & 1 \\ -u_0^2 + P'(\rho_0) & 2u_0 \end{bmatrix}.$$

This can be written out as (2.47):

$$\tilde{\rho}_t + (\widetilde{\rho u})_x = 0$$

 $(\widetilde{\rho u})_t + (-u_0^2 + P'(\rho_0))\widetilde{\rho}_x + 2u_0(\widetilde{\rho u})_x = 0.$

Rewrite in terms of p and u perturbations (Exer. 2.1): $\tilde{p}_t + u_0 \tilde{p}_x + K_0 \tilde{u}_x = 0,$

 $\rho_0 \tilde{u}_t + \tilde{p}_x + \rho_0 u_0 \tilde{u}_x = 0,$

where $K_0 = \rho_0 P'(\rho_0)$ is the bulk modulus.

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Linearization of gas dynamics

$$\tilde{p}_t + u_0 \tilde{p}_x + K_0 \tilde{u}_x = 0,$$

$$\rho_0 \tilde{u}_t + \tilde{p}_x + \rho_0 u_0 \tilde{u}_x = 0,$$

gives the system $q_t + Aq_x = 0$ (Drop tildes)

$$q(x,t) = \begin{bmatrix} p(x,t) \\ u(x,t) \end{bmatrix}, \qquad A = \begin{bmatrix} u_0 & K_0 \\ 1/\rho_0 & u_0 \end{bmatrix}$$

Eigenvalues: $\lambda = u_0 \pm c_0$

where $c_0 = \sqrt{K_0/\rho_0} = \sqrt{P'(\rho_0)}$ is the linearized sound speed.

Usually $u_0 = 0$ for linear acoustics. Then $\lambda^1 = -c_0$, $\lambda^2 = +c_0$.

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Example: Linear acoustics in a 1d tube

$$q = \left[\begin{array}{c} p \\ u \end{array} \right] \qquad \begin{array}{c} p(x,t) = \text{pressure perturbation} \\ u(x,t) = \text{velocity} \end{array}$$

Equations:

$$\begin{array}{lll} p_t + \kappa u_x &= 0 & \qquad \kappa &= \text{bulk modulus} \\ \rho u_t + p_x &= 0 & \qquad \rho &= \text{density} \end{array}$$

or

$$\left[\begin{array}{c} p \\ u \end{array}\right]_t + \left[\begin{array}{cc} 0 & \kappa \\ 1/\rho & 0 \end{array}\right] \left[\begin{array}{c} p \\ u \end{array}\right]_x = 0.$$

Eigenvalues: $\lambda = \pm c$, where $c = \sqrt{\kappa/\rho} = \text{sound speed}$

Second order form: Can combine equations to obtain

$$p_{tt} = c^2 p_{xx}$$

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Riemann Problem

Special initial data:

$$q(x,0) = \begin{cases} q_l & \text{if } x < 0 \\ q_r & \text{if } x > 0 \end{cases}$$

Example: Acoustics with bursting diaphram



Pressure:



Acoustic waves propagate with speeds $\pm c$.

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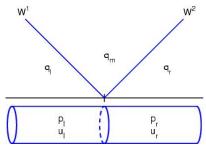
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Riemann Problem for acoustics

Waves propagating in x-t space:



Left-going wave $\mathcal{W}^1=q_m-q_l$ and right-going wave $\mathcal{W}^2=q_r-q_m$ are eigenvectors of A.

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Eigenvectors for acoustics

$$A = \left[\begin{array}{cc} u_0 & K_0 \\ 1/\rho_0 & u_0 \end{array} \right]$$

Eigenvectors:

$$r^1 = \left[\begin{array}{c} -\rho_0 c_0 \\ 1 \end{array} \right], \qquad r^2 = \left[\begin{array}{c} \rho_0 c_0 \\ 1 \end{array} \right].$$

Check that $Ar^p = \lambda^p r^p$, where

$$\lambda^1 = u_0 - c_0, \qquad \lambda^2 = u_0 + c_0.$$

with
$$c_0 = \sqrt{K_0/\rho_0} \implies K_0 = \rho_0 c_0^2$$
.

Note: Eigenvectors are independent of u_0 .

Let $Z_0 = \rho_0 c_0 = \sqrt{K_0 \rho_0} = \text{impedance}.$

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Diagonalization of linear system

Consider constant coefficient linear system $q_t + Aq_x = 0$.

Suppose hyperbolic:

- Real eigenvalues $\lambda^1 \leq \lambda^2 \leq \cdots \leq \lambda^m$,
- Linearly independent eigenvalues r^1, r^2, \ldots, r^m .

Let $R = [r^1 | r^2 | \cdots | r^m]$ $m \times m$ matrix of eigenvectors.

Then $Ar^p=\lambda^p r^p$ means that $AR=R\Lambda$ where

$$\Lambda = \left[\begin{array}{ccc} \lambda^1 & & & \\ & \lambda^2 & & \\ & & \ddots & \\ & & & \lambda^m \end{array} \right] \equiv \operatorname{diag}(\lambda^1, \lambda^2, \dots, \lambda^m).$$

 $AR = R\Lambda \implies A = R\Lambda R^{-1} \quad \text{and} \quad R^{-1}AR = \Lambda.$ Similarity transformation with R diagonalizes A.

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Diagonalization of linear system

Consider constant coefficient linear system $q_t + Aq_x = 0$.

Multiply system by R^{-1} :

$$R^{-1}q_t(x,t) + R^{-1}Aq_x(x,t) = 0.$$

Introduce $RR^{-1} = I$:

$$R^{-1}q_t(x,t) + R^{-1}ARR^{-1}q_x(x,t) = 0.$$

Use $R^{-1}AR = \Lambda$ and define $w(x,t) = R^{-1}q(x,t)$:

$$w_t(x,t) + \Lambda w_x(x,t) = 0$$
. Since R is constant!

This decouples to m independent scalar advection equations:

$$w_t^p(x,t) + \lambda^p w_x^p(x,t) = 0.$$
 $p = 1, 2, ..., m.$

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Solution to Cauchy problem

Suppose $q(x,0) = \overset{\circ}{q}(x)$ for $-\infty < x < \infty$.

From this initial data we can compute data

$$\overset{\circ}{w}(x) \equiv R^{-1}\overset{\circ}{q}(x)$$

The solution to the decoupled equation $w_t^p + \lambda^p w_x^p = 0$ is

$$w^{p}(x,t) = w^{p}(x - \lambda^{p}t, 0) = \overset{\circ}{w}^{p}(x - \lambda^{p}t).$$

Putting these together in vector gives w(x,t) and finally

$$q(x,t) = Rw(x,t).$$

We can rewrite this as

$$q(x,t) = \sum_{p=1}^{m} w^{p}(x,t) r^{p} = \sum_{p=1}^{m} \overset{\circ}{w}(x - \lambda^{p}t) r^{p}$$

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Physical meaning of eigenvectors

Eigenvectors for acoustics:

$$r^1 = \left[\begin{array}{c} -\rho_0 c_0 \\ 1 \end{array} \right] = \left[\begin{array}{c} -Z_0 \\ 1 \end{array} \right], \qquad r^2 = \left[\begin{array}{c} \rho_0 c_0 \\ 1 \end{array} \right] = \left[\begin{array}{c} Z_0 \\ 1 \end{array} \right].$$

Consider a pure 1-wave (simple wave), at speed $\lambda^1 = -c_0$, If $\overset{\circ}{q}(x) = \bar{q} + \overset{\circ}{w}^{1}(x)r^{1}$ then

$$q(x,t) = \bar{q} + \overset{\circ}{w}^{1}(x - \lambda^{1}t)r^{1}$$

Variation of q, as measured by q_x or $\Delta q = q(x + \Delta x) - q(x)$ is proportional to eigenvector r^1 , e.g.

$$q_x(x,t) = \mathring{w}_x^1(x - \lambda^1 t)r^1$$

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Physical meaning of eigenvectors

Eigenvectors for acoustics:

$$r^1 = \left[\begin{array}{c} -\rho_0 c_0 \\ 1 \end{array} \right] = \left[\begin{array}{c} -Z_0 \\ 1 \end{array} \right], \qquad r^2 = \left[\begin{array}{c} \rho_0 c_0 \\ 1 \end{array} \right] = \left[\begin{array}{c} Z_0 \\ 1 \end{array} \right].$$

In a simple 1-wave (propagating at speed $\lambda^1 = -c_0$),

$$\left[\begin{array}{c} p_x \\ u_x \end{array}\right] = \beta(x) \left[\begin{array}{c} -Z_0 \\ 1 \end{array}\right]$$

The pressure variation is $-Z_0$ times the velocity variation.

Similarly, in a simple 2-wave ($\lambda^2 = c_0$),

$$\left[\begin{array}{c} p_x \\ u_x \end{array}\right] = \beta(x) \left[\begin{array}{c} Z_0 \\ 1 \end{array}\right]$$

The pressure variation is Z_0 times the velocity variation.

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Acoustic waves

$$q(x,0) = \begin{bmatrix} \stackrel{\circ}{p}(x) \\ 0 \end{bmatrix} = -\frac{\stackrel{\circ}{p}(x)}{2Z_0} \begin{bmatrix} -Z_0 \\ 1 \end{bmatrix} + \frac{\stackrel{\circ}{p}(x)}{2Z_0} \begin{bmatrix} Z_0 \\ 1 \end{bmatrix}$$

$$= w^1(x,0)r^1 + w^2(x,0)r^2$$

$$= \begin{bmatrix} \stackrel{\circ}{p}(x)/2 \\ -\stackrel{\circ}{p}(x)/(2Z_0) \end{bmatrix} + \begin{bmatrix} \stackrel{\circ}{p}(x)/2 \\ \stackrel{\circ}{p}(x)/(2Z_0) \end{bmatrix}.$$

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Solution by tracing back on characteristics

The general solution for acoustics:

$$\begin{split} q(x,t) &= w^1(x-\lambda^1 t,0) r^1 + w^2(x-\lambda^2 t,0) r^2 \\ &= w^1(x+c_0t,0) r^1 + w^2(x-c_0t,0) r^2 \end{split}$$

Recall that $w(x,0) = R^{-1}q(x,0)$, i.e.

$$w^{1}(x,0) = \ell^{1}q(x,0), \qquad w^{2}(x,0) = \ell^{2}q(x,0)$$

where ℓ^1 and ℓ^2 are rows of R^{-1} .

$$R^{-1} = \left[\begin{array}{c} \ell^1 \\ \ell^2 \end{array} \right]$$

Note: ℓ^1 and ℓ^2 are left-eigenvectors of A:

$$\ell^p A = \lambda^p \ell^p$$
 since $R^{-1} A = \Lambda R^{-1}$.

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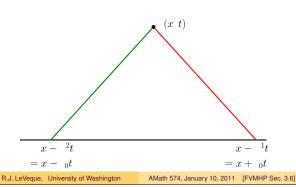
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Solution by tracing back on characteristics

The general solution for acoustics:

$$q(x,t) = w^{1}(x - \lambda^{1}t, 0)r^{1} + w^{2}(x - \lambda^{2}t, 0)r^{2}$$
$$= w^{1}(x + c_{0}t, 0)r^{1} + w^{2}(x - c_{0}t, 0)r^{2}$$



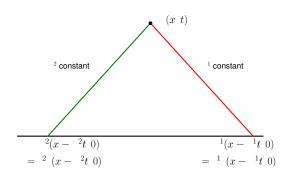
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Solution by tracing back on characteristics

The general solution for acoustics:

$$q(x,t) = w^{1}(x - \lambda^{1}t, 0)r^{1} + w^{2}(x - \lambda^{2}t, 0)r^{2}$$



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