

## CEE 342 Aut 2005 HW#8 Solutions

8.7. For laminar flow in a pipe or system with a similar flow geometry, flow is laminar at Reynolds numbers  $< 2100$ . At  $10^\circ\text{C}$ , the kinematic viscosity ( $\nu$ ) of water is  $1.31 \times 10^{-6} \text{ m}^2/\text{s}$  (from Table B.2, p.761). The velocity of the fluid is:

$$V = \frac{Q}{A} = \frac{(4 \text{ cm}^3/\text{s})}{\pi(0.4 \text{ cm})^2/4} = 31.8 \frac{\text{cm}}{\text{s}}$$

The Reynolds number is therefore:

$$\text{Re} = \frac{DV}{\nu} = \frac{(0.4 \text{ cm})(31.8 \text{ cm/s})}{(1.31 \times 10^{-6} \text{ m}^2/\text{s})(100 \text{ cm/m})^2} = 971$$

The flow is laminar. For laminar flow, the ratio of the entrance length to the diameter is given by:

$$\frac{l_e}{D} = 0.06 \text{Re} = 0.06(971) = 58.3$$

$$l_e = 58.3D = 58.3(0.4 \text{ cm}) = 23.3 \text{ cm}$$

Since the straw length (25 cm) is longer than  $l_e$ , the flow is fully developed.

8.15. The maximum pressure is the pressure that causes the Reynolds number to equal 2100. Thus:

$$\text{Re} = 2100 = \frac{DV\rho}{\mu}$$

$$V = \frac{2100\mu}{D\rho} = \frac{2100(0.30 \text{ N}\cdot\text{s}/\text{m}^2)}{(0.1 \text{ m})(10^3 \text{ kg}/\text{m}^3) \left(1 \frac{\text{N}}{\text{kg}\cdot\text{m}/\text{s}^2}\right)} = 6.3 \frac{\text{m}}{\text{s}}$$

For laminar flow, the headloss and pressure change due to friction are given by the Hagen-Poiseuille equation. If, in addition to friction, pressure changes because of a change in elevation, the overall pressure change is given by Equation 8.11 or 8.12. Using Equation 8.11, we find:

$$V = \frac{(\Delta p - \gamma l \sin \theta) D^2}{32\mu l}$$

$$\Delta p = \frac{32\mu l V}{D^2} + \gamma l \sin \theta = \frac{32(0.30 \text{ N}\cdot\text{s}/\text{m}^2)(10 \text{ m})(6.3 \text{ m/s})}{(0.10 \text{ m})^2} + \left(9810 \frac{\text{N}}{\text{m}^3}\right)(10 \text{ m}) \sin(-90^\circ)$$

$$= -37,620 \text{ Pa} = -37.6 \text{ kPa}$$

8.19. The velocity profile for laminar flow in a pipe is given by:

$$V = V_c \left(1 - \left[\frac{r}{R}\right]^2\right)$$

where  $V_c$  is the centerline velocity,  $r$  is the distance measured from the centerline, and  $R$  is the pipe radius. In the current case,  $R = 0.05 \text{ cm}$ ,  $r = (0.050 - 0.012) \text{ cm}$ , or  $0.038 \text{ cm}$ , and  $V = 0.8 \text{ m/s}$ . Also, the average velocity for laminar flow in a pipe is one-half of the maximum (centerline) velocity. The centerline velocity and flow are therefore:

$$V_c = \frac{V}{1 - (r/R)^2} = \frac{0.8 \text{ m/s}}{1 - (0.038/0.05)^2} = 1.89 \text{ m/s}$$

$$Q = V_{avg} A = 0.5 V_c A = (0.5)(1.89 \text{ m/s}) \left(\pi (0.05 \text{ m})^2\right) = 7.42 \times 10^{-3} \frac{\text{m}^3}{\text{s}}$$

8.21. The Darcy-Weisbach equation applies for either laminar or turbulent flow. Thus, for either case of interest:

$$h_L = f \frac{l V^2}{D 2g}$$

Since  $l$ ,  $D$ ,  $V$  and  $g$  are the same for the two cases, the ratio of the headloss in laminar to turbulent flow is the ratio of the corresponding friction factors. For laminar flow,  $f = 64/\text{Re}$ ; for  $\text{Re} = 6000$ ,  $f = 0.0107$ . For turbulent flow with a smooth pipe, we can find the friction factor from the Moody diagram, with the result that, for  $\varepsilon/D = 0$  and  $\text{Re} = 6000$ ,  $f = 0.035$ . Therefore, if the flow were laminar, the headloss would be reduced by:

$$\frac{0.035 - 0.0107}{0.035} = 0.694 = 69.4\%$$