

1 Flow-noise and turbulence in two tidal channels

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3 Christopher Bassett

4 Department of Mechanical Engineering, University of Washington, Seattle, Stevens Way, Box
5 352600, Seattle, WA 98165

6

7 Jim Thomson

8 Applied Physics Laboratory, University of Washington, Seattle, 1013 NE 40th St., Box 355640,
9 Seattle, WA 98105-6698

10 Peter Dahl

11 Applied Physics Laboratory, University of Washington, Seattle, 1013 NE 40th St., Box 355640,
12 Seattle, WA 98105-6698

13 Brian Polagye

14 Department of Mechanical Engineering, University of Washington, Seattle, Stevens Way, Box
15 352600, Seattle, WA 98165

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17 **1 Abstract**

18 Flow-noise resulting from oceanic turbulence and interactions with pressure-sensitive transduc-
19 ers can interfere with ambient noise measurements. This noise source is particularly important
20 in low-frequency measurements (< 100 Hz) and in highly turbulent environments such as tidal
21 channels. This work presents measurements made in the Chacao Channel, Chile and in Admi-
22 ralty Inlet, Puget Sound, Washington. In both environments, peak currents exceed 3 m s^{-1} and
23 pressure spectral densities attributed to flow-noise are observed at frequencies up to 500 Hz.
24 At 20 Hz, flow-noise exceeds mean slack noise levels by more than 50 dB. Two semi-empirical
25 flow-noise models are developed and applied to predict flow-noise at frequencies from 20-500 Hz
26 using measurements of current velocity and turbulence. The first model directly applies mean
27 velocity and turbulence spectra while the second model relies on scaling arguments that relate
28 turbulent dissipation to the mean velocity. Both models, based on prior formulations for infra-
29 sonic ($f < 20$ Hz) flow-noise, agree well with observations in Chacao Channel. In Admiralty
30 Inlet, good agreement is shown only with the model that applies mean velocity and turbulence
31 spectra, as the measured turbulence violates the scaling assumption in the second model.

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35 **2 Introduction**

36 Pressure fluctuations occur when fluid moves relative to an immersed body at high Reynolds
37 numbers. If sufficiently large in magnitude, these fluctuations can be measured by a pressure
38 sensitive transducer. This phenomenon, called *flow-noise* or *pseudosound*, is a result of both
39 advected ambient turbulence and interactions with the transducer in the flow. Unlike other
40 ambient noise sources, flow-noise does not propagate and should not be included in ambient
41 noise statistics. In some cases, the magnitude of these pressure fluctuations is much greater
42 than those associated with ambient noise. Although flow-noise is fundamentally a low frequency
43 phenomenon, the range of frequencies over which flow-noise can interfere with ambient noise
44 measurements is dependent on the intensity of turbulence and the transducer geometry.

45 Flow-noise in the ocean has been identified in a number of applications utilizing stationary
46 measurement platforms. In Narragansett Bay, Rhode Island measurements in the octave band
47 centered at 25 Hz (Willis and Dietz, 1965) and frequency bands from 40 to 100 Hz (Dietz *et al.*,
48 1960) were found to be strongly correlated with tidal cycles. Webb (1988) identifies infrasonic
49 flow-noise in the bottom boundary layer. Flow-noise induced by wave orbital motion has been
50 reported up to 500 Hz (Gobat and Grosenbaugh, 1997), and flow-noise on moored instruments
51 in shallow water has been reported at frequencies below 50 Hz (Deane, 2000).

52 While flow-noise often appears in measurements, predicting its acoustic spectrum is a more
53 difficult task. Webb (1988) and Strasberg (1979, 1985) identify and suggest models for infrasonic
54 flow-noise based largely on scaling arguments related to the mean flow velocity. Beyond the
55 infrasonic range ($f < 20$ Hz), there are no generalized models for flow-noise. For advected
56 turbulence with a length scale that is small in comparison to the characteristic size of the
57 hydrophone, phase variations across the surface of the hydrophone cause pressure fluctuations
58 to partially cancel and increase the spectral slope of flow-noise relative to turbulence (Strasberg,

59 1979). This study focuses on flow-noise associated with turbulent scales that are similar to and
60 smaller than the size of the hydrophone.

61 There are few published ambient noise studies in highly energetic environments, defined here
62 as locations where turbulent kinetic energy (TKE) exceeds $0.0025 \text{ m}^2 \text{ s}^{-2}$ and mean currents
63 in exceed of 0.5 m s^{-1} . In recent years, an interest exploiting strong currents for tidal power
64 generation has led to ambient noise studies in such locations to understand the potential envi-
65 ronmental impacts of power production (e.g., Polagye et al. (in revision)). One of the primary
66 environmental concerns associated with commercial-scale tidal power extraction is the acoustic
67 emissions from tidal turbines because of their potential to affect marine mammal behavior (Po-
68 lagye *et al.*, 2011). At tidal energy sites, flow-noise presents a significant challenge to quantifying
69 ambient noise, especially at low frequencies.

70 Drifting platforms are one approach to characterizing sound in high-flow environments since
71 the relative velocity between drifters and the mean flow is small. This limits both the advection
72 of turbulence across the transducer and turbulence shed by the transducer. However, drifter
73 studies are labor intensive, convolve space and time, and cannot cost-effectively quantify tem-
74 poral variations in sound over long time scales (e.g., hours to months). Therefore, a robust
75 assessment of ambient noise conditions or turbine noise requires that instrumentation packages
76 must be deployed over time periods longer than permitted by drifting studies, and thus must
77 be exposed to strong currents.

78 **2.1 Turbulent velocity fluctuations**

79 Currents in tidal channels have velocity components related to deterministic tides, meteorolo-
80 gical currents (e.g., storm surges and wave-induced currents), and turbulence (Polagye and
81 Thomson, 2013). If meteorological currents are negligible, the current velocity can be repre-

82 sented as $u = \bar{u} + u'$, where \bar{u} is the mean, deterministic velocity which is nearly constant over
 83 short (e.g., 5-10 minute) sample periods and u' are the turbulent velocity fluctuations. In gen-
 84 eral, u is a vector with along-channel, cross-channel, and vertical components when projected
 85 along the principal axis. Hereafter, all velocity references refer to the component projected
 86 along the principal axis of the flow.

87 Identification of the range of temporal and spatial scales of turbulence is critical to the
 88 interpretation of fluctuations sensed by a pressure sensitive transducer. Previous work in high-
 89 current environments identifies two important turbulent domains: the large-scale, horizontal
 90 eddies containing most of the turbulent kinetic energy, and the small-scale isotropic eddies of
 91 the inertial subrange (Thomson *et al.*, 2012). Turbulent kinetic energy is dissipated at smaller
 92 length scales (dissipation range). Kolmogorov (1941) provides the theoretical basis for the
 93 assumption of isotropic turbulence and an analysis of measurable spatial patterns in isotropic
 94 turbulence. The theoretical wavenumber spectrum in the inertial subrange is written as

$$S_u(k) = a\epsilon^{2/3}k^{-5/3}, \quad (1)$$

95 where $S_u(k)$ is the turbulent spectrum, ϵ is the dissipation rate of turbulent kinetic energy, a is
 96 a constant, and k is the spatial wavenumber of the turbulence scales. Invoking Taylor's frozen
 97 turbulence hypothesis (Taylor, 1938) and substituting the frequency for the wavenumber, the
 98 turbulence spectrum in the inertial subrange is described by

$$S_u(f) = a\epsilon^{2/3}f^{-5/3} \left(\frac{\bar{u}}{2\pi} \right)^{2/3}, \quad (2)$$

99 where f is the frequency and \bar{u} is the mean advected velocity.

100 The turbulent energy of large-scale, typically anisotropic, eddies is transferred to smaller
 101 scales, a process that continues until viscous forces damp out the fluctuations. Basic scaling

102 arguments are used to identify the spatial and temporal scales at which turbulent fluctuations
103 occur. The smallest turbulent scales found in the inertial subrange, the Kolmogorov microscales,
104 are related to the viscosity and the dissipation rate of turbulent kinetic energy. The Kolmogorov
105 length scale (η_o) is defined as

$$\eta_o = \left(\frac{\nu^3}{\epsilon} \right)^{1/4}, \quad (3)$$

106 where ν is the kinematic viscosity. At frequencies greater than those associated with the Kol-
107 mogorov microscales the spectral slope is steeper than in the inertial subrange due to preferential
108 damping by viscosity.

109 **2.2 Turbulent pressure fluctuations**

110 Turbulence in the ocean can generate noise through multiple mechanisms: sound radiated by
111 turbulence and noise resulting from the presence of a pressure sensitive transducer in the flow.
112 Proudman (1952) and Lighthill (1952, 1954) provide a theoretical basis for the radiation of sound
113 by turbulence. The relationships developed in these papers highlight that the sound radiation
114 efficiency has a strong dependence on the Mach number (M^5), defined as \bar{u}/c , where \bar{u} is the
115 advected velocity and c is the celerity. Given the celerity of seawater ($\approx 1500 \text{ m s}^{-1}$), the Mach
116 number is small and sound generated by turbulence underwater is radiated inefficiently. Noise
117 levels attributed to this radiated sound are well below typical ambient noise levels in the ocean
118 (Wenz, 1962; Ross, 1976).

119 The second mechanism, pressure fluctuations associated with turbulence, cannot be mea-
120 sured remotely because they do not produce propagating sound waves. The non-propagating
121 pressure fluctuations that can be locally measured are advected turbulent pressure fluctua-
122 tions and fluctuations resulting from the interaction between the sensor and the turbulent flow

123 (Strasberg, 1979, 1985; Webb, 1988). In their analyses, Strasberg (1979, 1985) and Webb (1988)
 124 identify spectra associated with flow-noise at low frequencies (< 20 Hz) in relatively low-velocity
 125 conditions ($\bar{u} < 0.5$ m s $^{-1}$) without co-temporal turbulence data. These studies serve as the
 126 starting point for this analysis. In the absence of interactions between pressure fluctuations and
 127 the sensor, the wide-band pressure fluctuations are related to the velocity fluctuations according
 128 to

$$\overline{p^2} = \rho^2 \left(\overline{u'^2} \right)^2, \quad (4)$$

129 where $\overline{p^2}$ is the mean-square pressure fluctuation, ρ is the ambient density, and $\overline{u'^2}$ is the mean-
 130 square along-channel velocity fluctuation (Kraichnan, 1956). Strasberg (1979) hypothesized a
 131 relationship between velocity and pressure spectra,

$$S_p(f) = \rho^2 \overline{u'^2} S_u(f), \quad (5)$$

132 where $S_p(f)$ is the pressure variance spectrum and $S_u(f)$ is the velocity variance spectrum.
 133 While Eq. 4 can be obtained by integrating over all frequencies in Eq. 5, as Strasberg (1979)
 134 notes, there is no theoretical basis for this relationship.

135 Strasberg (1979, 1985) related frequency-dependent pressure fluctuations to velocity fluctu-
 136 ations according to

$$S_p(f) = \rho^2 \bar{u}^2 S_u(f), \quad (6)$$

137 where the turbulent velocity term in Eq. 5 is replaced with the mean velocity. When velocity
 138 fluctuations are advected across the sensor, the observed frequency is related to mean velocity
 139 and wavelength of the velocity fluctuation (λ) according to $f = \bar{u}/\lambda$. It is expected that a
 140 sensor in the flow will be most sensitive to turbulent scales that exceed the largest dimension

141 (d) of the sensor such that $fd/\bar{u} \ll 1$ is satisfied (i.e., the turbulent “gust” engulfs the entire
142 sensor). In this regime, the observed trend (e.g., slope) of the turbulent velocity spectrum
143 is expected to be equivalent to that in the pressure spectrum. In this study, the frequencies
144 under consideration exceed 20 Hz and it will be shown that the associated spatial scales do not
145 satisfy $fd/\bar{u} \ll 1$. Therefore, attenuation of the pressure signal due to partial cancelation of
146 the pressure fluctuations is expected, which increases the slope of the spectrum in this regime.
147 Analysis of flush-mounted hydrophones suggests the attenuation scales with the dimensionless
148 frequency according to $(fd/\bar{u})^n$ and n is related to the hydrophone geometry (Urlick, 1975).

149 This paper presents velocity and noise data obtained in two energetic tidal channels: the
150 Chacao Channel, Chile and northern Admiralty Inlet, Puget Sound, Washington. At both sites,
151 peak tidal currents exceed 3 m s^{-1} , and these strong currents produce significant flow-noise
152 for moored transducers. Co-temporal turbulence and noise measurements from the Chacao
153 Channel are used to develop models for flow-noise and extend previous analyses of flow-noise to
154 higher frequencies. These models are then compared to observations from Admiralty Inlet. The
155 following sections discuss the theoretical principles and results of the flow-noise models. Sec. 3
156 includes a description of the measurement sites, moorings, data acquisition systems, and signal
157 processing methods. Sec. 4 presents the data from both sites and Sec. 5 interprets the results
158 and discusses the implications for studies in high velocity environments.

159 **3 Methods**

160 **3.1 Sites**

161 Co-temporal hydrodynamic and acoustic measurements were obtained from two locations in this
162 study: the Chacao Channel near Carelmapu, Chile and Admiralty Inlet near Port Townsend,

163 Washington, USA. Because of large dynamic range of velocities at both sites, there are no
164 periods with true “slack” currents throughout the water column. Here “slack” refers to periods
165 when $\bar{u} < 0.3 \text{ m s}^{-1}$ at the depth of the sensor platforms.

166 In Chacao Channel, Chile a mooring, referred to as the Tidal Turbulence and Acoustics
167 Mooring (TTAM), was deployed to obtain hydrodynamic and acoustic measurements at S 41°
168 45.75', W 73° 40.95' from February 11, 2013 to February 14, 2013 (Fig. 1a-b). The mooring was
169 deployed at a depth of approximately 38 m from the R/V Dr. Jurgen Winter, a research vessel
170 operated by the Universidad Austral de Chile.

171 By using a mooring instead of a rigid platform deployed on the seabed, measurements were
172 taken outside of the region in which boundary layer effects are most significant without utilizing
173 a prohibitively large and expensive platform. The TTAM, further described in Thomson *et al.*
174 (2013), consisted of three major components: a heavy anchor ($\approx 1050 \text{ kg}$ in water) to hold the
175 mooring in place, a vane on which the instruments were mounted, and a 94 cm float ($\approx 317 \text{ kg}$
176 of buoyancy) to hold the mooring line in tension.

177 The vane for mounting the instrumentation was deployed in-line between the anchor and the
178 float 9 m above the seabed at slack water. Swivels mounted to both ends of the vane provided a
179 passive yawing mechanism to keep the instruments aligned into the principal axis of the flow and
180 minimized the risk of mooring components interfering with the sensors. Two acoustic Doppler
181 velocimeters (ADVs) and two autonomous hydrophone packages were deployed on the vane.
182 The dual-package approach provided redundancy and maintained symmetry across the vane.

183 The Chacao Channel, in comparison with many coastal environments in the United States,
184 has relatively small amounts of commercial shipping traffic. Shipping vessels and cruise ships
185 bound for Puerto Montt occasionally transited the site and three large vessels were noted during
186 daytime hours throughout the deployment period. While large shipping traffic may operate

187 without considering the local currents, local fishing patterns are determined by them. Local
188 fishers leave the village prior to slack tide to dive for 40 minutes near the deployment site before
189 retuning. The local fishing vessels were, in general, less than 10 m long and have operating
190 air compressors for surface-supplied diving. With the exception of overnight slack tides, the
191 acoustic signals of small vessels appeared in nearly all of the slack tide recordings.

192 During slack water, the spectrum levels increase to approximately 80 dB re $1\mu\text{Pa}^2 \text{Hz}^{-1}$ at
193 5 kHz, which is consistent with noise from snapping shrimp (Everest *et al.*, 1948; Readhead,
194 1997). Above 1 kHz, there are also increases in noise levels that diverge from the expected
195 flow-noise spectrum. Energetic sites like these with beds composed of grains smaller than 10 cm
196 can produce significant amounts of sediment-generated noise (Bassett *et al.*, 2013). The data
197 to confirm this noise source are not available; however, the observed increases at frequencies
198 greater than 1 kHz during non-slack conditions are consistent with the noise that would be
199 produced by a mobilized bed composed of pebbles and gravel (Thorne, 1985, 1986).

200 In Admiralty Inlet, USA, co-temporal velocity and noise measurements were obtained using
201 autonomous instrumentation packages on a tripod deployed from the R/V Jack Robertson at
202 $48^\circ 09.120' \text{N}$, $122^\circ 41.152' \text{W}$ (depth 55 m) from February 11-21, 2011 (Fig. 1c-d). In this
203 area, Admiralty Inlet is about 5 km wide and shipping lanes and local ferry traffic result in high
204 densities of vessel traffic (Bassett *et al.*, 2012). This site has been the subject of acoustic studies
205 including vessel noise (Bassett *et al.*, 2012) and sediment-generated noise (Bassett *et al.*, 2013)
206 in addition to turbulence studies (Thomson *et al.*, 2012; Thomson *et al.*, 2013).

207 An Oceanscience (www.oceanscience.com) Sea Spider tripod with additional lead ballast
208 was modified to allow for the deployment of a variety of instrumentation packages and recovery
209 floats. The instruments relevant to this study, an ADV and two hydrophones, were mounted
210 vertically on one side of the tripod. The sampling volume of the ADV and the hydrophones were

211 vertically aligned such that they each recorded 1.05 m from the seabed. There was a horizontal
212 separation distance of approximately 30 cm between the hydrophones. The ADV was deployed
213 such that the sampling volume was located at the midpoint between the hydrophones.

214 **3.2 Turbulence and mean velocity measurements**

215 The ADVs used to measure mean current velocities and turbulence at both sites were 6 MHz
216 Nortek Vectors. In the Chacao Channel, the ADVs sampled at 16 Hz for 300 sec every 10 min.
217 Velocity spectra were calculated for 128 sec time windows with 50% overlap. In Admiralty
218 Inlet the sampling frequency was 32 Hz for 256 sec every 10 min and spectra were calculated
219 using 32 sec time windows with 50% overlap. In the case of the Chacao Channel, an x-IMU
220 (inertial motion unit) was mounted within the ADV pressure housing. IMU data was used to
221 remove mooring motion contamination from the velocity spectra (Thomson *et al.*, 2013). For
222 both data sets, each sample period were projected on to the principal axis, reviewed for quality,
223 and despiked using the phase-space method (Goring and Nikora, 2002; Mori *et al.*, 2007). ADV
224 data were used to calculate turbulence spectra, mean velocities, and the dissipation rate. Mean
225 current velocities were linearly interpolated to form a times series with 1-min resolution.

226 **3.3 Acoustic measurements**

227 In both the Chacao Channel and Admiralty Inlet, the autonomous hydrophones were Loggerhead
228 Instruments DSG data acquisition systems equipped with Hi-Tech (HTI-96-MIN) hydrophones
229 approximately 1.9 cm in diameter and 5 cm long. In the Chacao Channel, the hydrophones were
230 deployed on the mooring vane at a -20° angle relative to the horizontal such that the change of
231 angle due to drag on the mooring resulted in the hydrophones being oriented roughly lengthwise
232 into the along-channel flow. Throughout the deployment, the systems recorded ambient noise

233 continuously with a sampling frequency of 80 kHz. Spectra were calculated using windows
234 with 65 636 data points (≈ 0.8 sec) and a 50% overlap. Each window was tapered using a
235 Hann window. The resulting spectra had a frequency resolution of $\Delta f = 1.22$ Hz and were
236 truncated to only include frequencies between 0.020-25 kHz (low frequency linear limit to the
237 hydrophone through maximum frequencies of interest). Given that each recording was 62.5 sec
238 long, timestamps were rounded down to the nearest minute for comparison to ADV data.

239 The motion and vibrations of the mooring caused by strong currents and turbulence con-
240 tributed to self-noise. Using a combination of manual review of the audio and visual inspection
241 of spectrograms, the signatures of two types of self-noise produced by the mooring were identi-
242 fied. The first type, clanging and creaking sounds associated with floats, shackles, and mooring
243 vane, occurred occasionally. The most easily identifiable peaks attributed to this self-noise oc-
244 curred around 700 Hz although energy was present between 300-1000 Hz (Fig. 2). Even during
245 noisy periods, these sounds were identifiable.

246 The second type mooring noise, attributed to mooring vibrations, appeared in spectrograms
247 as continuous noise when current velocities exceeded 0.9 m s^{-1} . In individual recordings, these
248 peaks appeared as a constant, lower intensity noise. Examples of these peaks are visible in Fig.
249 6. In the spectrum associated with current velocities from $0.9\text{-}1.2 \text{ m s}^{-1}$ there was a notable
250 peak around 90 Hz that shifts to higher frequencies with increases in current velocity. These
251 peaks in the spectra were relatively narrow and adjacent frequency bands are consistent with
252 the characteristics of flow-noise.

253 These noises were not removed from the recordings; instead, further analysis focuses on the
254 characteristics of the spectra outside of these frequency bands. More specifically, the analysis
255 of flow-noise was limited to frequencies below 500 Hz and those that didn't have peaks that
256 regularly diverged from the expected spectrum of flow-noise. It should also be noted that

257 these increases were not consistent with vortex shedding from the hydrophone. For example,
258 the Strouhal number for associated with a cylinder in such flows is approximately 0.2 (Schewe,
259 1983), which implies a shedding frequency of 12 Hz during peak currents, was too low to explain
260 the 90 Hz noise.

261 Signals from vessel traffic in the Chacao Channel was, at times, easily identifiable, partic-
262 ularly during slack currents. The sound produced by local fishing vessels, cruise ships, and
263 shipping vessels resulted in increased noise levels across a broad range of frequencies. To re-
264 move these points from the data set, all data were manually reviewed for signals consistent with
265 vessel noise. Such signal characteristics included a series of tones below 500 Hz and broadband
266 increases up to 20 kHz that evolved on time scales consistent with the passage of a vessel – up
267 to 30 minutes depending on the vessel type (Greene and Moore, 1995; McKenna *et al.*, 2012;
268 Bassett *et al.*, 2012). These recordings were flagged and not included in the analysis of flow-
269 noise. A total of 466 recordings out of a total 2959 were removed due to obvious vessel traffic.
270 Although some of the retained recordings likely include some degree of vessel traffic, this was
271 limited to signals with relatively low pressure spectral densities.

272 The same hydrophones and processing techniques were used in Admiralty Inlet with two
273 exceptions: the hydrophones recorded 10 sec at the top of every minute and they were deployed
274 such that the hydrophones elements were oriented vertically in the water column. Admiralty
275 Inlet has been previously identified as a noisy environment in which vessel traffic is a significant
276 contributor to ambient noise levels (Bassett *et al.*, 2012). To identify vessel traffic, an Auto-
277 matic Identification System (AIS) receiver located near the site was used to record real-time
278 vessel traffic. In post-processing, times when an AIS transmitting vessel was within 10 km
279 were identified and removed from the data set. The exclusion of all instances when an AIS
280 transmitting vessel was present within the detection range of the AIS system (≈ 20 km) would

281 have resulted in few data points to analyze. In addition, many small vessels do not transmit
282 AIS data. Unlike the Chacao Channel data, manual review of the data was not used to remove
283 vessel traffic not identified by the AIS system due to the challenges of identifying these signals
284 in a noisy environment.

285 **3.4 Flow-noise models**

286 Two models relating the flow-noise pressure spectra to current velocity and turbulence are
287 developed using observations at the Chacao Channel site, which has more favorable measurement
288 conditions (i.e., less vessel traffic). The two models are based on the frequency-dependent
289 relationship between velocity and pressure (Eq. 6) as given by Strasberg (1979, 1985). The
290 first model estimates flow-noise from measured mean velocity and turbulence, while the second
291 model estimates flow-noise on the basis of mean velocity alone.

292 Eq. 5 also provides a basis for the development of a model; the only difference between Eqs.
293 5 and 6 is that the first contains the mean-square velocity fluctuation term while the second
294 contains a mean-velocity squared term. This results in a difference of approximately 3 orders of
295 magnitude between the two models. Given the agreement between observed data and models
296 based on Eq. 6, the models based on Eq. 5 are not of primary importance. The development of
297 these models in the following sections refer forward, by necessity, to some of the results formally
298 presented in Sec. 4.

299 **3.4.1 Flow-noise model 1: pressure spectra derived from turbulence measurements**

The first model, referred to as the “turbulence model”, relates the pressure spectra to the mean velocity and turbulence spectra. A semi-empirical approach is applied to scales where the assumption that the $fd/\bar{u} \ll 1$, where d is the size of the transducer, is violated. This

approach requires the quantification two terms empirically: the slope of the observed pressure spectra due to partial cancelation of pressure fluctuations over the surface of the hydrophone (m) and the frequencies at which the change in slope due to decreased sensitivity occurs. The frequency at which the transition occurs is referred to as the shoulder frequency (f_{sh}). Through the substitution of the turbulence spectrum (Eq. 2) into the relationship for the pressure and velocity spectra (Eq. 6), the velocity spectrum is converted to a pressure spectrum based on the scales of the turbulence according to

$$\tilde{S}_p(f) = \begin{cases} a\rho^2\bar{u}^2\epsilon^{2/3}f^{-5/3}\left(\frac{\bar{u}}{2\pi}\right)^{2/3}, & \text{if } f < f_{sh} \\ a\rho^2\bar{u}^2\epsilon^{2/3}f^{-m}\left(\frac{\bar{u}}{2\pi}\right)^{2/3}, & \text{if } f > f_{sh} \end{cases} \quad (7)$$

where the tilde denotes the model.

To determine the empirical constants (f_{sh} and m) the slopes of the observed pressure spectra are calculated from 30-70 Hz. These frequencies are chosen because below 30 Hz, the frequency response of the recording system begins to roll-off and above 70 Hz mooring self-noise and lower signal-to-noise ratios (flow-noise being the signal) reduce the method accuracy. These fits are further discussed in Sec. 4.2, but for the purpose of model development we note that the observed slope follows $f^{-3.2}$, thus $m = -3.2$. This slope is applied to Eq. 8 and the shoulder frequency is identified by iteratively adjusting this term to maximize agreement with the observed pressure spectra. The best relationship between the modeled and observed spectra occurs when

$$f_{sh} = 0.1 \left(\frac{\bar{u}}{d}\right). \quad (9)$$

In other words, the sensitivity to turbulent scales decreases due to partial cancellation if these scales not at least 10 times larger than the characteristic size of the hydrophone. In this formulation, the size of the hydrophone that is applied in both models is length of the hydrophone in the direction of the along-channel flow (i.e., 5 cm in Chacao Channel and 1.9 cm in Admiralty

313 Inlet). The shoulder frequency is always less than the frequencies included in this study (< 20
 314 Hz). Therefore, further development of the flow-noise model is limited to Eq. 8.

315 Modeled spectrum levels (in dB re $1\mu\text{Pa}^2 \text{ Hz}^{-1}$) are given by

$$\tilde{S}_p(f) = 10 \log_{10} \left(\frac{a\rho^2\bar{u}^2\epsilon^{2/3}f_{sh}^{-5/3}\left(\frac{\bar{u}}{2\pi}\right)^{2/3}\left(\frac{f}{f_{sh}}\right)^{-3.2}}{10^{-12}} \right), \quad \text{if } f > f_{sh} \quad (10)$$

316 and the modeled pressure spectra are obtained by applying the dissipation rate, mean velocity,
 317 and water density (1024 kg m^{-3}). The modeled spectra are calculated for the same frequencies
 318 as the measured acoustic spectra.

319 3.4.2 Flow-noise model 2: pressure spectra derived from mean-flow

One major limitation in the development of the turbulence model is that it requires an estimate for the dissipation rate, which is more difficult to obtain in high flow environments than the mean velocity. As an alternative, turbulence scaling arguments are used to develop a flow-noise model, referred to as the “mean-flow model,” based on the mean advected velocity and the slope of the pressure spectra. The first scaling argument relates the dissipation rate to the largest scales of turbulence in a flow. An estimate of the rate of transfer of energy to turbulence yields $\epsilon \propto u_{rms}^3/l$, where u_{rms} is the root-mean-square of the velocity and l is the scale of the energy containing scales (Lumley, 1983; Thorpe, 2007). The velocity is decomposed as $u = \bar{u} + u'$ and the turbulence intensity is defined as

$$I = \sigma_{u'}/\bar{u}, \quad (11)$$

where $\sigma_{u'}$ is the standard deviation. If $I^2 \ll 1$, then by the mathematical definition root-mean-square velocity, $u_{rms} \approx \bar{u}$ and

$$\epsilon \propto \frac{\bar{u}^3}{l}. \quad (12)$$

320 Thomson *et al.* (2012) showed that the dominant scales of the TKE in two well-mixed, energetic
 321 tidal channels to be well represented by three times the water depth, assuming the tidal elevation
 322 is small compared to the depth. Therefore, the eddy scale l in the Chacao Channel is assumed
 323 to be constant so that $\epsilon \propto \bar{u}^3$. As shown in Sec. 4.1, the observed dissipation rates are consistent
 324 with this scaling at the Chacao Channel site.

325 Assuming that the hydrophone-specific attenuation factor $(fd/\bar{u})^n$, combined with Eq. 6, is
 326 able to describe the flow-noise, it follows that the observed velocity and frequencies dependencies
 327 should be satisfied. Therefore, the pressure spectrum scales as

$$\tilde{S}_p(f) = \rho^2 \bar{u}^2 S_u(f) \left(\frac{fd}{\bar{u}} \right)^n. \quad (13)$$

328 By combining Eqs. 2 and 6 and substituting \bar{u}^3 for ϵ , the pressure spectrum becomes

$$\tilde{S}_p(f) = a \rho^2 \bar{u}^{14/3} f^{-5/3} \left(\frac{fd}{\bar{u}} \right)^n, \quad (14)$$

329 where the water depth (l) is rolled into the constant a . This is an underdetermined system in
 330 which both the \bar{u} and f dependencies must be satisfied by n . A slope of $f^{-3.2}$ is identified,
 331 resulting in $n \approx -1.5$, a value that agrees well with the observed velocity dependence in the
 332 Chacao Channel. The final formulation (in $\text{Pa}^2 \text{Hz}^{-1}$) based on this semi-empirical approach is

$$\tilde{S}_p(f) = c \rho^2 \bar{u}^{6.1} f^{-3.2}, \quad (15)$$

333 where c is a constant which now represents a combination of the constant from the power fit
 334 scaling between the dissipation rate and \bar{u} (including the length scale l), the prior constant (a),
 335 and the slope transition associated with the sensitivity to turbulence of different scales. Notably,
 336 the magnitude of this scalar offset is dependent, in part, on the mechanisms responsible for the
 337 production and dissipation of turbulence, which vary significantly in natural environments.

338 Because of this, the model’s framework may be applicable to other sites where $\epsilon \propto \bar{u}^3$, but the
 339 magnitude is expected to vary. To simplify the analysis and demonstrate the effectiveness of
 340 this method, the results from Eq. 15 are regressed against the observed noise levels to identify
 341 this constant. The final, spectral form (in dB re $1\mu\text{Pa}^2 \text{ Hz}^{-1}$) is

$$\tilde{S}_p(f) = b + 10 \log_{10} \left(\frac{\rho^2 \bar{u}^{6.1} f^{-3.2}}{10^{-12}} \right), \quad (16)$$

342 where b is a constant.

343 4 Results

344 4.1 Turbulence results

345 The measurements of turbulence and mean currents made during five-minute periods are affected
 346 by two types of mooring motion. The first, a change in the angle of the mooring line due to
 347 drag forces on the float, is related to the mean currents. During peak currents, 40° angles
 348 relative to the vertical are observed, resulting in a measurement position approximately 6 m
 349 above the bed in comparison to 9 m above the bed during slack currents (Thomson *et al.*, 2013).
 350 However, the change occurs over time scales longer than 5 minutes and along-channel currents
 351 may be obtained by correcting for orientation from the internal ADV sensor. The second type
 352 of mooring motion is caused by turbulence, and can contaminate the velocity measurements.
 353 Thomson *et al.* (2013) includes analysis of the same velocity data set and demonstrates that
 354 data can be post-processed to obtain accurate turbulence spectra, as shown on a mean-velocity
 355 bin-averaged basis in Fig. 3. The inertial subrange for isotropic turbulence, which has a $f^{-5/3}$
 356 dependence and is used to determine the dissipation rate of TKE, is evident in all of the bin-
 357 averaged spectra. The $f^{-5/3}$ slope continues until viscous dissipation damps out the turbulent

358 fluctuations and the flattening of the spectra at high frequencies in Fig. 3 is caused by the
359 lower limit of the instrument’s sensitivity. The peaks in the TKE spectra during low current
360 periods are caused by low-frequency mooring oscillations. When currents exceed 0.6 m s^{-1} the
361 amplitude of the peaks in the spectra decreases due to increased tension in the mooring line.

362 Consistent with scaling arguments, the dissipation rate of turbulent kinetic energy increases
363 with the mean current velocity. Fig. 4 includes time series data for the mean current velocity
364 and the dissipation rate. The observed current velocities range from 0 to 3 m s^{-1} on both flood
365 and ebb tides. The dissipation rates, which are shown to scale with \bar{u}^3 (Fig. 5a), range from
366 less than $10^{-5} \text{ m}^2 \text{ s}^{-3}$ around slack water to greater than $10^{-3} \text{ m}^2 \text{ s}^{-3}$ during peak currents.
367 While the dissipation rate scales with \bar{u}^3 at this site, it should be noted that the scaling of
368 turbulence is largely dependent on local features of the flow (e.g., bathymetry) so this result
369 cannot be assumed to apply in other tidal channels (Thomson *et al.*, 2013). A time series of
370 the theoretical maximum frequency of turbulent pressure fluctuations in the inertial subrange
371 (Eq. 3), also included in Fig. 4c, shows microscale frequencies ranging from 100 Hz during slack
372 currents to greater than 10 kHz during peak currents.

373 The mean current velocity and turbulence measurements in Admiralty Inlet, just as in the
374 Chacao Channel, reveal a highly energetic environment with peak observed near-bed current
375 velocities of approximately 2 m s^{-1} . Unlike the measurements in the Chacao Channel, the
376 measurements in Admiralty Inlet are within the bottom boundary layer where a large velocity
377 gradient is observed (Bassett *et al.*, 2013) and the dissipation rates do not scale with \bar{u}^3 over
378 the entire range of \bar{u} (Fig. 5b).

379 4.2 Acoustic results

380 In the Chacao Channel, slack current noise levels are relatively flat below 500 Hz with spectrum
381 levels between 60-70 dB re $1 \mu\text{Pa}^2 \text{Hz}^{-1}$. These low current conditions provide the baseline for
382 analyzing increases due to flow-noise. Fig. 6 includes average spectra in 0.3 m s^{-1} velocity bins.
383 Once current velocities reach approximately 0.3 m s^{-1} an acoustic signature consistent with flow-
384 noise begins to dominate low-frequency measurements. At 20 Hz the difference between slack
385 current noise levels and those produced by flow-noise are 10 dB, and increases are observed
386 up to 100 Hz. As currents increase, the noise levels at the lowest frequencies do so as well,
387 reaching levels up to 70 dB above those during slack currents. At the same time, flow-noise is
388 observed over a wider range of frequencies. A simple analysis based on when observed noise levels
389 diverge from the expected “red” spectrum of flow-noise suggests that it is likely to mask other
390 sound sources at frequencies up to 800 Hz during the strongest currents. Both the velocity-bin
391 averaged spectra and individual spectra show that the spectral slope of the flow-noise follows
392 a $f^{-3.2}$ dependence. As previously mentioned, the theoretical maximum frequency at which
393 flow-noise could be measured, given these dissipation rates, exceeds 10 kHz at the site during
394 peak currents. Although flow-noise is not observed at frequencies greater than 1 kHz at this
395 site, it is reasonable to speculate that at a site with less other noise above 1 kHz – for example,
396 a site without sediments (i.e., scoured bedrock) and no snapping shrimp – flow-noise could be
397 observed at higher frequencies.

398 The flow-noise observed in Admiralty Inlet is comparable to that observed in the Chacao
399 Channel. Fig. 8a includes bin-averaged noise spectra up to 200 Hz in Admiralty Inlet. During
400 slack tide periods, observed spectrum levels are approximately 80 dB re $1 \mu\text{Pa}^2 \text{Hz}^{-1}$. Flow-noise
401 is measured at frequencies less than 30 Hz once currents exceed 0.3 m s^{-1} and at frequencies
402 greater than 200 Hz during peak currents. Peak flow-noise levels exceed slack tide conditions

403 by up to 50 dB at 20 Hz. The observed slopes of the pressure spectra in Admiralty Inlet are in
404 agreement with those the Chacao Channel.

405 **4.2.1 Model-data comparisons**

406 In the Chacao Channel, the magnitude of the constant (b) used in the mean-flow model (Eq. 16)
407 is 48 dB re $1\mu\text{Pa}^2 \text{ Hz}^{-1}$. Physically, this constant combines both the scalar constant from the
408 dissipation scaling and the shoulder frequency (f_{sh}). The contribution of the scalar constant for
409 the dissipation scaling, calculated as $20\log_{10}(1.4 \cdot 10^{-4}/10^{-6})$, yields an offset of 43 dB. Using a
410 typical f_{sh} of 3 Hz, a value associated with current velocities of 1.5 m s^{-1} , the spectrum shift
411 is calculated as $10\log_{10}(20/3)$, or 8 dB. This term can be thought of as a shift necessary to
412 account for neglecting the consideration of the hydrophone’s sensitivity to different turbulent
413 scales in the initial model. Combining these terms leads to an offset of 51 dB, a value close to
414 the 48 dB offset shown in Fig. 7.

415 A comparison of nine representative, observed pressure spectra at current velocities covering
416 the entire dynamic range in the Chacao Channel versus modeled pressure spectra using both
417 models are included in Fig. 7, which demonstrates that both models agree well with observed
418 spectra over most current velocities at the site (i.e., the spectra cover current velocities from 0.3-
419 2.7 m s^{-1}). With the exception of the 0.3 and 0.6 m s^{-1} spectra, which diverge from the expected
420 behavior of flow noise above 100 and 200 Hz, respectively, the models show good agreement up
421 to 500 Hz. With the exception of frequencies at which self-noise from the mooring is identified,
422 the model typically agrees with observations to within a few decibels.

423 Also included in Fig. 7 are scatter plots of observed pressure spectral densities in the Chacao
424 Channel versus modeled pressure spectral densities at five frequencies between 30 and 480 Hz.
425 Again, generally good agreement is found between the models and the observations. In the case

426 of the higher frequencies (240 and 480 Hz) measurements associated with the weakest currents
427 are cut-off by the y-axis. In this regime there is poor agreement between the observations and
428 models because flow-noise does not have a significant impact at these frequencies during low
429 current periods. At these same frequencies, there is also less agreement during peak currents
430 (highest pressure spectral densities), with observations always exceeding the models. These
431 differences, which can also be seen in Figs. 7a and 7b, are also attributed to mooring noise. To
432 quantify the performance of both models, R^2 values for the modeled versus observed spectra
433 level for the five frequencies in Fig. 7 are determined and included in Table I. The R^2 values
434 for both models range from 0.95 for both models at 30 Hz to less than 0.7 at 480 Hz. The
435 decreased R^2 values at higher frequencies are attributed to lower signal-to-noise ratios at higher
436 frequencies.

437 Fig. 8b includes a comparison between observed and modeled flow-noise levels in Admiralty
438 Inlet using the turbulence model at three frequencies: 30 Hz, 60 Hz, and 120 Hz. There is good
439 agreement between the model and observations at these frequencies but there is more scatter
440 than the results for the Chacao Channel. The additional scatter is attributed, in part, to the
441 shorter length of the recordings in Admiralty Inlet, which results in pressure spectra with larger
442 uncertainties. Additionally, the acoustic recordings are much shorter than the averaging periods
443 for the turbulence spectra and are not sufficiently long to record throughout the entire period
444 of the largest turbulent scales at the site. In comparison to the Chacao Channel data, there is
445 also significantly more scatter in the estimated dissipation rates at a given velocity in Admiralty
446 Inlet (Fig. 5). Disagreement at the lowest noise levels occurs during periods of low current when
447 flow-noise levels are below ambient levels. Other significant outliers are attributed to other
448 sources such as vessels without AIS transponders (such as military traffic).

449 The mean-flow model is not a good predictor of flow-noise in Admiralty Inlet. A compar-

450 ison of the observations and the mean-flow model is not included due to the poor agreement.
451 Specifically, the observed flow-noise increases at a more rapid rate than predicted by the model.
452 This is not surprising given the dissipation rate increases more rapidly than \bar{u}^3 as shown in Fig.
453 5b. Therefore, a critical assumption in the mean-flow model is not valid at the site.

454 5 Discussion

455 As shown in Sec. 4.2, it is necessary to consider flow-noise when quantifying low-frequency am-
456 bient noise levels in areas characterized by high turbulence and currents. While the geographic
457 distribution of such environments is quite limited, ongoing development in these areas, as well
458 as their importance as biological choke points, will likely lead to more acoustics studies prone
459 to high levels of flow-noise. For example, flow-noise will exceed received levels for close-range
460 vessel traffic (Bassett *et al.*, 2012) by 30 dB or more at frequencies less than 100 Hz.

461 To properly quantify propagating low-frequency noise in these environments using stationary,
462 autonomous platforms, the development of flow-noise mitigation techniques would be beneficial.
463 While signal processing using multiple transducers can be used to reduce or eliminate flow-noise
464 using cross-correlation techniques (Chung, 1977; Buck and Greene, 1980), these methods are
465 of little practical value when flow-noise exceeds ambient noise levels by more than 50 dB (i.e.,
466 signal-to-noise ratios are too low for signal processing to be effective). Flow-noise can also be
467 partially mitigated through the use of larger transducers, which will cause pressure fluctuations
468 to cancel more rapidly over the surface (Urick, 1975; Strasberg, 1979). However, due to the
469 large range of length scales of turbulence in energetic tidal channels, this may also be of limited
470 utility. Ultimately, this work demonstrates a need to develop devices that are compact (to
471 reduce drag) and shield transducers from the flow.

472 The models presented here, based on the basic formulations by Strasberg (1979, 1985),

473 show good agreement with the observed data. This data set spans a wider range of current
474 velocities than has been previously described in the literature, with peak currents nearly one
475 order of magnitude greater than those discussed in Strasberg (1979, 1985) and Webb (1988). In
476 addition, the frequencies studied here extend the regime of modeled flow-noise to well beyond
477 the infrasonic range. It is expected that different hydrophone aperture geometries would have
478 unique values for empirical constants (e.g., m , f_{sh}), although the same scaling relations would
479 apply. To facilitate the application of higher frequency flow-noise models, further research
480 regarding the response of hydrophones to small-scale pressure fluctuations is needed.

481 Both the turbulence and mean-flow models for flow-noise pressure spectra face certain lim-
482 itations in application to other sites. For example, the turbulence model requires estimates of
483 the dissipation rate or turbulence spectra in the inertial subrange. Ideally, turbulent kinetic
484 energy spectra would be measured in the frequency range overlapping with hydrophone mea-
485 surements to allow a direct evaluation of Eq. 6. However, the low-frequency response of the
486 hydrophones employed in this study did not overlap with the velocity measurements. As such,
487 extrapolation based on theoretical turbulence scaling was necessary. Nonetheless, the turbu-
488 lence model is shown to perform well in two energetic tidal channels. The mean-flow model,
489 by contrast, does not require turbulence measurements but relies on scaling arguments that are
490 not universally valid. In addition, the scalar constant for the mean-flow model is expected to
491 be site-specific. However, when these assumptions are valid, the scaling arguments can lead to
492 accurate predictions of flow-noise levels.

493 The turbulence model relates mean-velocity and dissipation rate measurements to observed
494 flow-noise levels. The good agreement between the model and measurements suggest that flow-
495 noise spectra could be used to estimate the dissipation rate given measurements of the mean
496 velocity when the response of the hydrophone (f_{sh} and m) is known.

497 Finally, the measurements presented here suffer from self-noise contamination attributed
498 to mooring noise. This is not surprising given the engineering challenges associated with the
499 development of a mooring to be deployed in such a dynamic environment. Isolating connection
500 points between different mooring components using noise dampening material is one method
501 of reducing the self-noise, which represents only one of many engineering challenges associated
502 with the acquisition of quality ambient noise measurements in such environments.

503 **6 Conclusions**

504 Measurements of ambient noise in the Chacao Channel, Chile, and Admiralty Inlet, Puget
505 Sound, Washington, two sites where peak currents exceed 3 m s^{-1} , reveal pressure spectral
506 densities attributed to flow-noise that exceed $135 \text{ dB re } \mu\text{Pa}^2 \text{ Hz}^{-1}$ at 20 Hz. These peak levels
507 can exceed ambient noise by more than 50 dB. Flow-noise is observed to frequencies greater
508 than 500 Hz. It is found that the slope of pressure spectra attributed to flow-noise is $f^{-3.2}$, a
509 value that is expected to depend on the geometry of the hydrophone.

510 Two semi-empirical models are presented for flow-noise that extend the frequency range of
511 prior models for infrasonic flow-noise. The first model utilizes direct measurements of turbulence
512 and mean current velocities while the second uses scaling arguments to model flow-noise only on
513 the basis of mean currents. Unlike other flow-noise studies, the frequency ranges considered here
514 are associated with scales of turbulence that are similar to and smaller than the hydrophone.
515 Both models are shown to agree well with flow-noise observations over the range of current
516 velocities observed in the Chacao Channel measurements. In the Admiralty Inlet measurements,
517 the turbulence model also agrees well with observations. However, the dissipation rate scales
518 differently with velocity and the mean-flow model performs poorly. The agreement between the
519 turbulence model and two observations in these dynamic channels extends the dynamic range

520 of flow-noise models by nearly an order of magnitude.

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Table I: R^2 values for both flow-noise models and the observed spectrum levels in the Chacao Channel at selected frequencies.

	Frequency (Hz)				
	30	60	120	240	480
Turbulence Model R^2	0.95	0.89	0.83	0.77	0.65
Mean-Flow Model R^2	0.95	0.87	0.83	0.80	0.68

Figure 1: (Color online) a-b). South-central Chile, the study area (red rectangle), and the deployment site (red circle). c-d). Puget Sound, the study area (red rectangle), and the deployment site (red circle).

Figure 2: A 40-second spectrogram from the Chacao Channel showing the transient signals associated with clanging and creaking noise centered around 700 Hz.

Figure 3: (Color online) Velocity-bin averaged (0.3 m s^{-1} bins) TKE spectra from the Chacao Channel show the expected $f^{-5/3}$ slope associated with isotropic turbulence. During periods with weak currents ($\bar{u} < 0.3 \text{ m s}^{-1}$), there is significant mooring motion contamination. However, during these periods flow-noise has little impact on observed ambient noise levels.

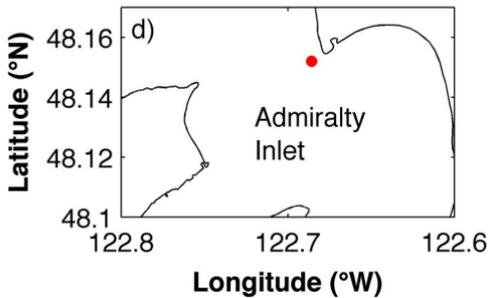
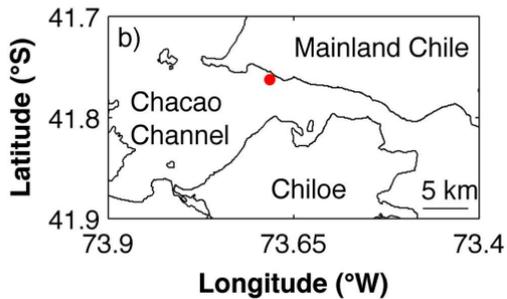
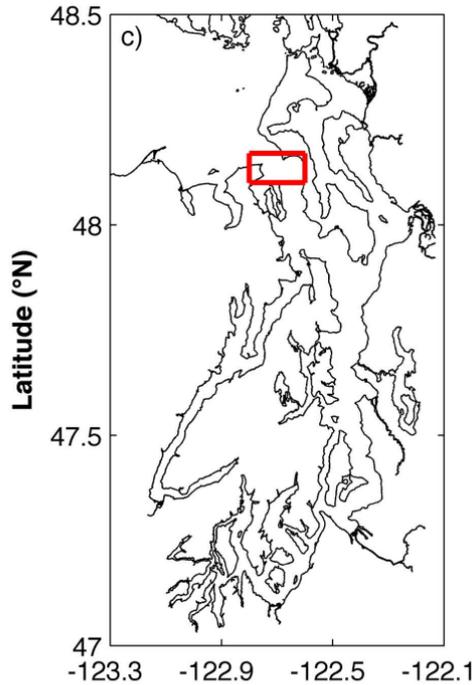
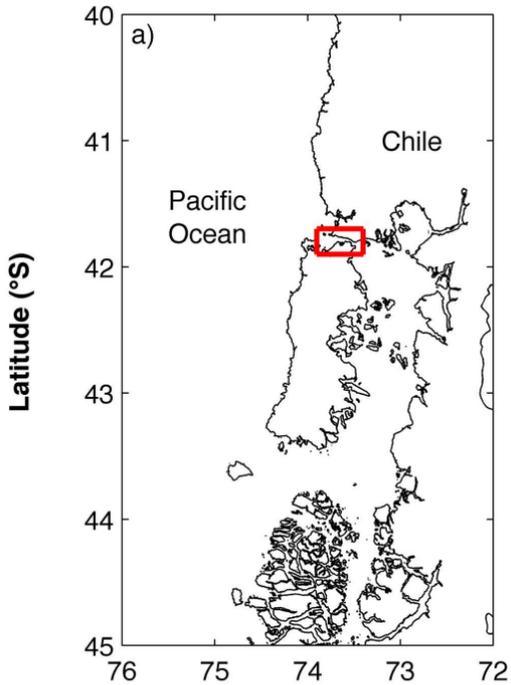
Figure 4: Hydrodynamics data from the Channel Channel. a). Time series of the mean current velocity. b). Time series of the dissipation rate of turbulent kinetic energy. c). Time series of the microscale frequency.

Figure 5: Scatter plots of the dissipation rate of turbulent kinetic energy versus current velocity in the Chacao Channel (a) and Admiralty Inlet (b). In Admiralty Inlet, the scaling used in the formulation of the mean-flow model ($\epsilon \sim \bar{u}^3$) is not valid.

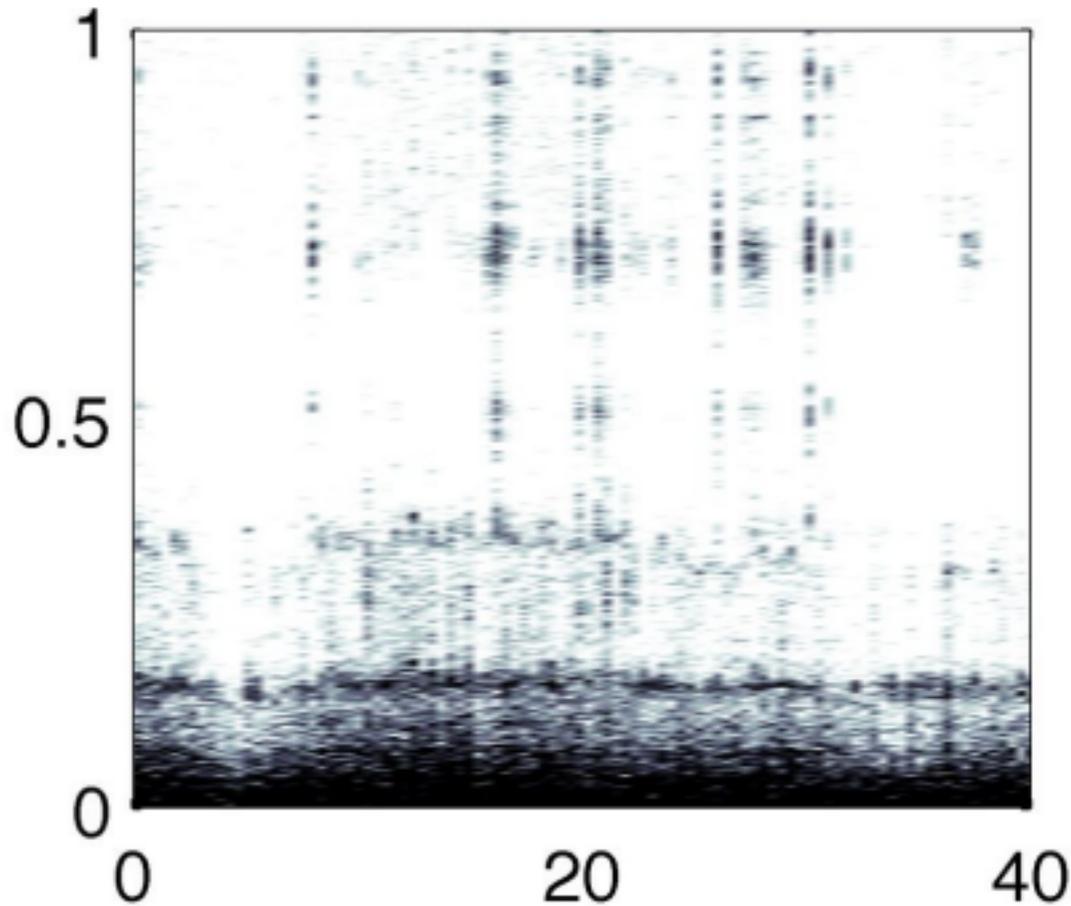
Figure 6: (color online) Velocity bin-averaged (0.3 m s^{-1} bins) pressure spectra from the Chacao Channel. The observed flow-noise has a $f^{-3.2}$ dependence.

Figure 7: (Color online) A comparison of observed and modeled flow-noise levels using the turbulence and mean-flow models in the Chacao Channel. a,b). Modeled and observed pressure spectra for both models. The black lines are the modeled spectra using the hydrodynamic measurements at the time of the hydrophone recordings. The range of current velocities is $0.3\text{-}2.7 \text{ m s}^{-1}$ in 0.3 m s^{-1} increments. The ten small insets includes comparison of observed and modeled pressure spectral densities at five frequencies for the turbulence and mean-flow models. The black lines are the show the 1-1 relationship.

Figure 8: (Color online) a). Velocity-bin averaged spectra (0.3 m s^{-1} velocity bins) in Admiralty Inlet for periods with no AIS transmitting vessels with 10 km of the hydrophone. The small insets include a comparison between the modeled and observed pressure spectra in Admiralty Inlet for the turbulence model at three frequencies. The mean-flow model is not shown because ϵ does not scale with \bar{u}^3 .



Frequency (kHz)



Time (sec)

PSD (dB re $1\mu\text{Pa}^2\text{ Hz}^{-1}$)

